

# SAE Baja '24 Capstone Team

## Presentation 1

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# Project Description



NAU SAE Baja 2020-2021

## What is SAE Baja?

The Society of Automotive Engineers (SAE) Baja Collegiate Design Series is an engineering challenge for students to design and build a single-seat, all-terrain vehicle.

- Compete against other universities
- 13 members total, 4 sub-teams
  - Front End, Rear End, Frame, Drivetrain
- Sponsors: Stay tuned!
- Successful performance puts NAU on the map, strengthens internal Baja knowledge, and grows NAU Baja industry sponsorship connections

# Background



*Baja ETS SOTA Vehicle*

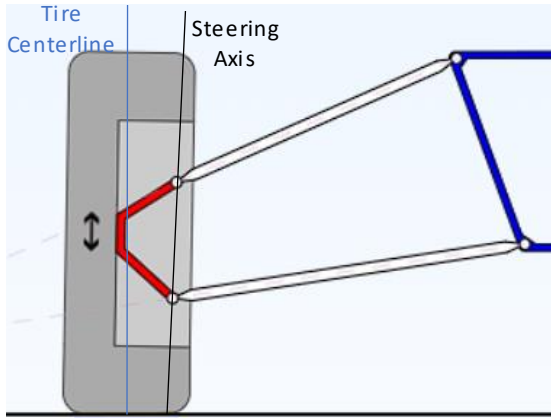
## “State-of-the-Art” University Teams

Creative innovation without rule violations is the goal. Some teams do this better than others due to funding & legacy knowledge:

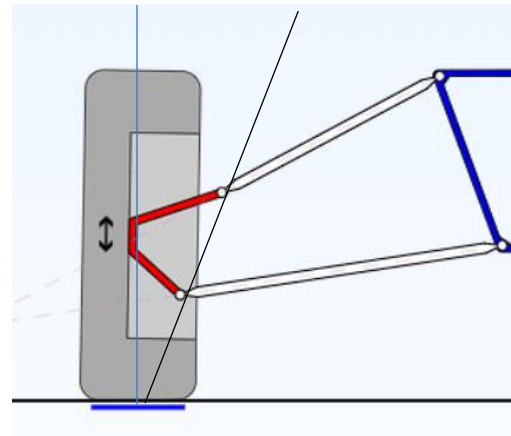
- Baja ETS
  - #1 Overall (Oregon 2023), Placed Top 3 in 8/10 Events
- Beaver Racing
  - #2 Overall (Oregon 2023), Place Top 3 in 4/10 Events
- Cornell Baja Racing
  - #3 Overall (Oregon 2023), Place Top 3 in 5/10 Events

# Benchmarking #1 – Front

Positive Scrub Radius



Zero Scrub Radius



## Knuckle Design - Scrub Radius

Many top teams go with a **zero (or near zero) scrub radius** when designing their front suspension. This minimizes scrub radius influence on steering and toe characteristics under braking/acceleration.

Cornell



Oregon State



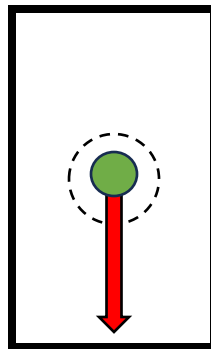
Contact Patch

Steering Axis

Moment Arm

Brake Force

Braking: Toe In



All forces kept in-line w/ steering axis!

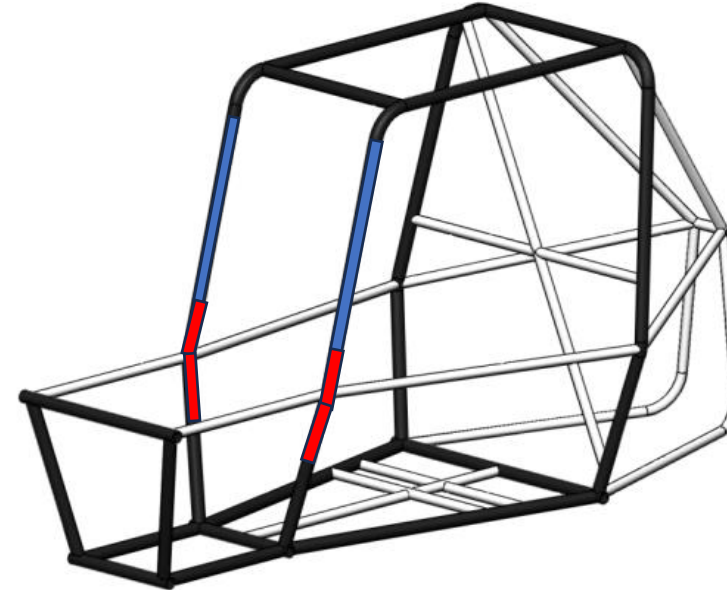
# Benchmarking #2 - Front

## Front Shock Placement

3 front shock lower mounting locations can be:

- Upper A arm mounted to the upper front brace member
- Upper A arm mounted to the lower front brace member
- Lower J arm mounted to the lower front brace member

- First Place 2023 Oregon: ETS – Lower front brace mounted
- First Place 2023 Ohio: CWRU – Lower front brace mounted
- First Place 2017 Oshkosh: UM Ann Arbor – Side impact/lower brace junction mounted



Upper Front Brace Members  
highlighted in blue

Lower Front Brace Member  
highlighted in red

# Benchmarking #3 - Front

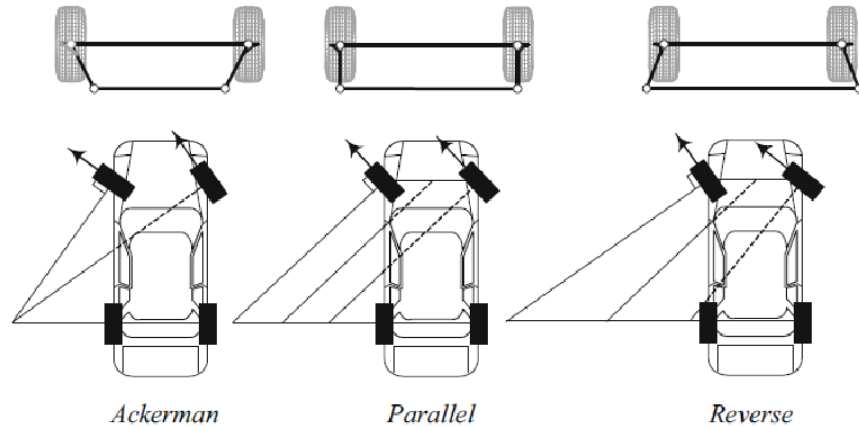


Diagram showing Ackermann, parallel, and Anti-Ackermann steering geometries. [24]



Cornell Ackermann Steering on Display

## Steering Design

Ackermann is most useful at very low speeds and tight turns because that is when you have the least wheel slip and load transfer side to side. (Gillespie, 1992) (Team, BYU Baja, 2017)

- Baja ETS - Ackermann
  - #1 Maneuverability (Oregon 2023)
- Cornell Baja Racing – 50% Ackermann
  - #2 Maneuverability (Oregon 2023)
- Beaver Racing - Ackermann
  - #3 Maneuverability (Oregon 2023)

# Customer & Engineering Requirements - Front

## Customer Requirements

- Vehicle must comply with the dimensions of SAE Baja courses
- Vehicle must have adequate ground clearance
- Vehicle must have adequate traction across all terrains
- Vehicle must be capable of safe operation over rough land terrain
- Vehicle must have agile maneuverability
- Front suspension must be robust in design (control arms, hubs, knuckles, tie rods, etc.)

## Engineering Requirements

- Decrease Vehicle Width
  - *Max Vehicle Width = 64"*
- Increase Ride Height
  - *Front Ride Height Minimum = 10"*
- Increase Tire Traction
  - *Scrub Radius = ~0 degrees*
- Increase Capability in Rough Terrain
  - *Wheel Travel = ~12" total (3:1 bump to droop)*
- Increase Turn-In Angle
  - *Pro-Ackerman = 40%-100%*
- Increase Crash Durability
  - *Target 40mph collision*

# QFD Discussion - Front

Customer Needs	Customer Weights	Decrease Vehicle Width	Increase Ride Height	Increase Tire Traction	Increase Capability in Rough Terrain	Increase Turn-In Angle	Increase Crash Durability	Customer Opinion Survey				
								1 Poor	2	3 Acceptable	4	5 Excellent
Comply with track dimensions	4	9									A	BC
Adequate ground clearance	2		9	6	9		3				A	C B
Adequate traction	3	3	3	9	6	3	3				A	BC
Safe operation over rough terrain	3	6	6	3	9		9					ABC
Agile maneuverability	4	6	3	6	3	9					A	BC
Robust design	3		3		3		9				BC	A
Technical Requirement Units		Inches	Inches	Degrees (Scrub Rad)	Inches (Wheel Travel)	Degrees	mph					
Technical Requirement Targets		64	10	0	12	40-100	40					
Absolute Technical Importance		1.87	5.66	3.72	2.84	6.45	4.69					
Relative Technical Importance		1	5	3	2	6	4					

Legend	
A	NAU #74
B	Baja ETS
C	Cornell Racing

## CR & ETSR Correlation

Many of the requirements from the rulebook work together with each other, leading to a more direct design path. The top 3 technical focuses for front end are as follows:

- **Modulating vehicle width** (track compliance)
- **Increasing rough terrain capability** (wheel travel)
- **Increasing tire traction** (scrub radius mitigation)

Other considerations include durability, ride height, and steering angle



# Literature Review

-Abraham Plis-

## Textbooks

- ***Suspension Geometry and Computation [3]***
  - Ch 12: Double Arm Suspensions
- ***The Automotive Chassis: Engineering Principles [4]***
  - Ch 1: Types of Suspension and Drive

## Papers

- ***Analysis of Steering Knuckle of All Terrain Vehicles (ATV) Using Finite Element Analysis [5]***
  - Knuckle Design/FEA
- ***Design and Development of Front Suspension System for an Off-Road Vehicle [6]***
  - A-Arm FEA & Anti-Dive Geometry
- ***Design Review of Suspension Assembly of a BAJA ATV [7]***
  - Lotus Shark Suspension Analysis

## Online

- ***Suspension & Steering Geometry (Front) | Double Wishbone | Anti-Ackerman | SAE BAJA | Solidworks [8]***
  - CAD Tutorial, A-Arm Geometry
- ***Steering Knuckle | Solidworks | 3D - Modelling | BAJA ATV [9]***
  - CAD Tutorial, Knuckle Design

# Literature Review

-Evan Kamp-

## Textbooks

- **Vehicle Dynamics: Theory and Application [23]**
  - Chapter 7: Steering Dynamics
  - Calculations for Turning Radius and Viable Steering Angles
- **The Science of Vehicle Dynamics Handling, Braking, and Ride of Road and Race Cars [77]**
  - Chapter 5: The Kinematics of Cornering

## Papers

- **Analysis of Ackermann Steering Geometry [74]**
- **Steering system for SAE Baja [46]**
  - Ackermann vs Parallel Steering Design
- **Design and Optimization of Steering Assembly for Baja ATV Vehicle [12]**
  - Design of Ackerman Steering Arms

## Online

- **Tech Explained: Ackermann Steering Geometry [59]**
  - Viable Steering Angles as a function of Slip Angle and Lateral Force
- ***Baja Virtual Presentation Series [79]***
  - *Day 8 - Steering Calculations*

# Literature Review

-Bryce Fennell-

## Papers

### ***Optimal Design of Suspension System of Four-Wheel Drive Baja Racing [34]***

- Suspension kinematics, shock mounting positional data

### ***Fine-Tuning Of the Suspension System of Baja ATV [33]***

- Camber angle, wheel axle path

### ***Redesigning the Cooper Union SAE Mini-Baja Front Suspension and Steering [35]***

- Steering angle, double a-arm suspension system, calculations

## Online

### ***Baja SAE, SAE International, 2023. [32]***

- Competition rules and guidelines for frame/suspension integration

### ***TUTORIAL ON LOTUS SUSPENSION SOFTWARE [31]***

- How to recover suspension data from the program

### ***Lotus Shark Suspension | Tutorial [30]***

- Basic overview of Lotus Shark software
- Suspension system input into program

## Textbooks

### ***Suspension Geometry and Computation [36]***

- Chapter 7, Camber and scrub
- Chapter 12, double-Arm Suspension

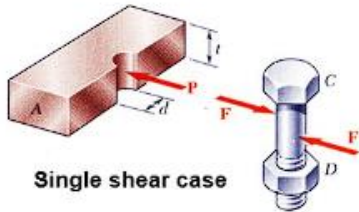
### ***Road and Offroad Vehicle Dynamics [58]***

- Suspension characteristics pg: 379-422

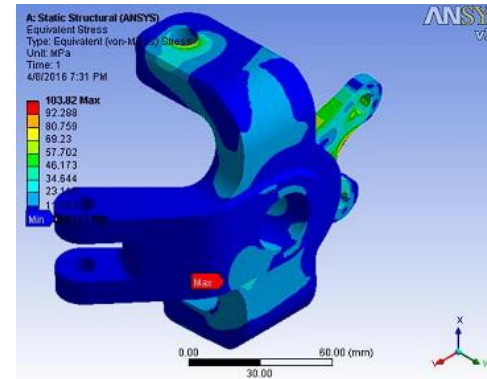
# Mathematical Modeling #1 - Front

## Equations, Tools, and Examples

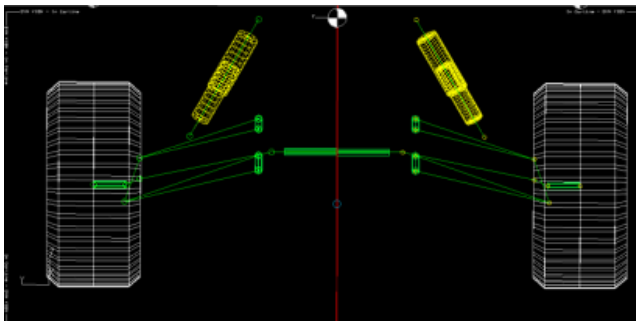
$$\sigma_b = \frac{P}{A_b} = \frac{P}{td}$$



Hole bearing stress equation for bolted connections to design against tear-out and deform [10]



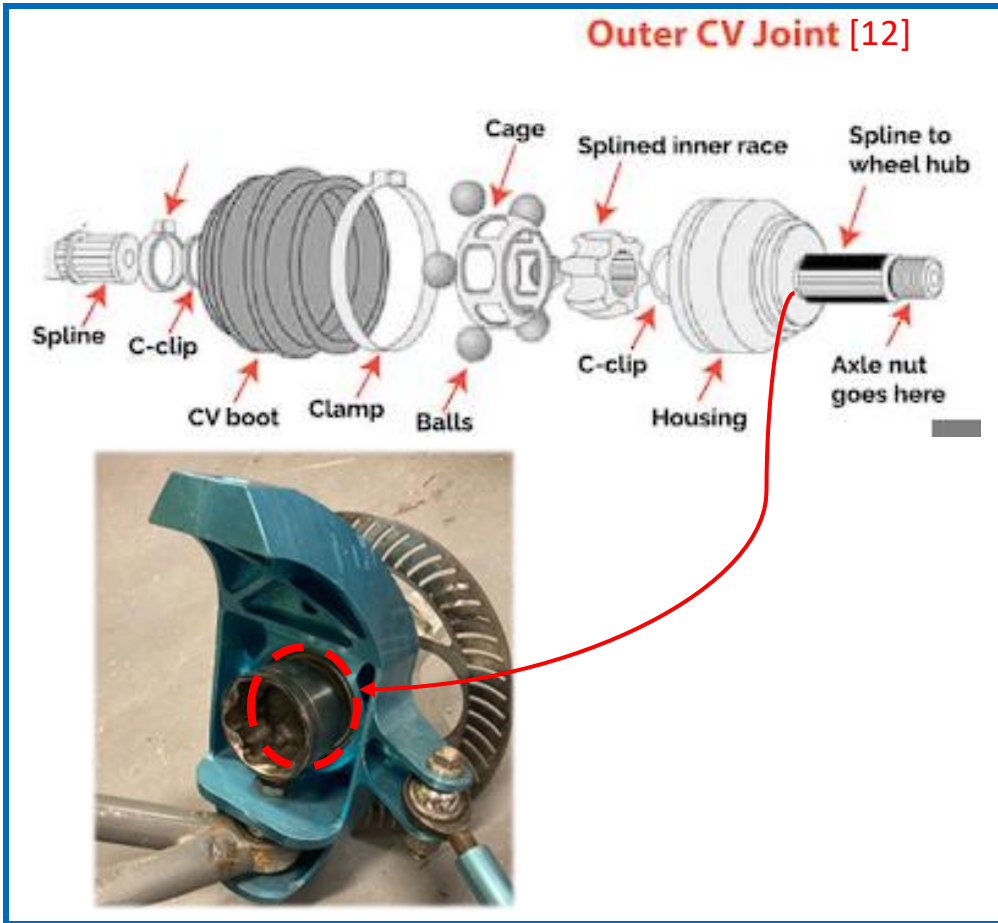
Excellent example of how to perform FEA analysis on a steering knuckle to optimize design characteristics



Lotus *Shark* software will help define and optimize suspension geometry, allowing hardpoints to be established [7]

# Mathematical Modeling #1 - Front

How much bearing stress will the central knuckle hole see from the outboard cv spline during a jump?



Standard Baja Weight (with driver): 500 lb.

Max Acceleration During Scenario: 3\*g

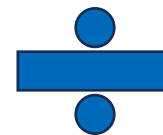
Drop Height: ~6 ft.

Duration of Impact: 0.1 seconds

Diameter of Spline: ~1"

Thickness of Contact Surface: ~2"

$$F = \frac{m \cdot \sqrt{g \cdot h}}{t} \quad [11]$$



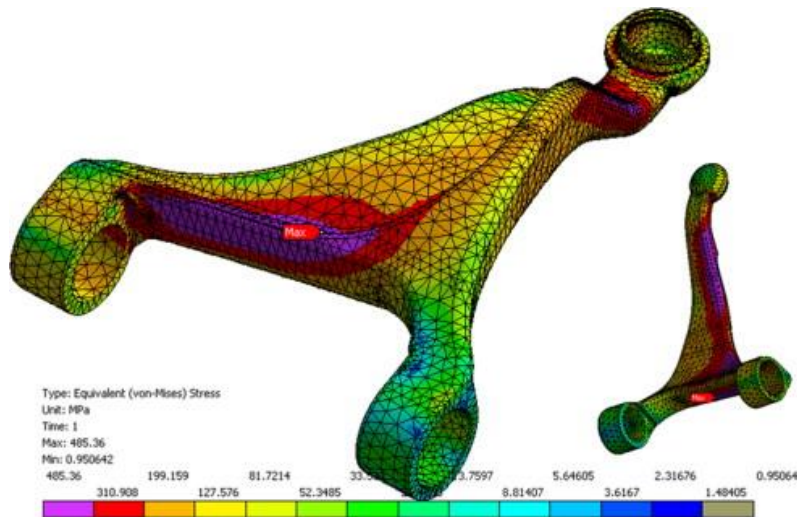
$$A_b = d \cdot t$$

$$\rightarrow \sigma_b = 1080 \text{ psi}$$

Answer can be validated by comparing to **yield strength** of billet aluminum (26,107 psi, meaning  $\sigma_b$  is 4% of that [13]). This tells us that **skeletonization** of the steering knuckle around the **inner contact surfaces is possible!**

# Mathematical Modeling #2 - Front

## A-Arm Design, Forces, and Inputs



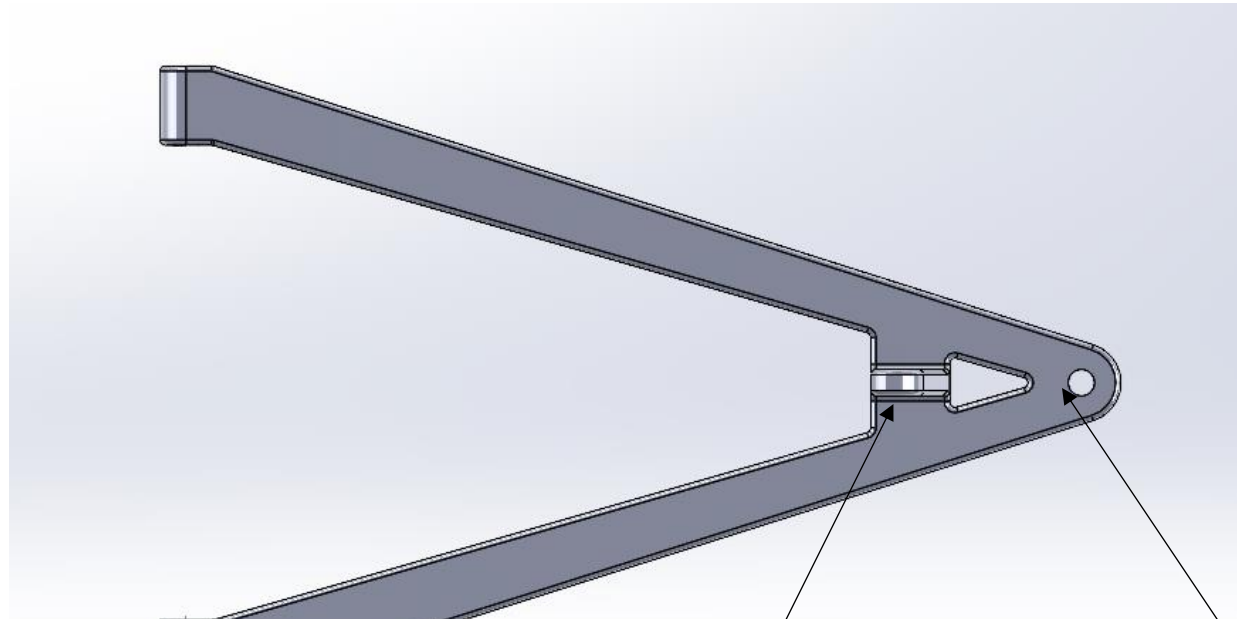
FEA of a modeled A-arm showing stresses developing under a compressive load



2023 ETS SAE Baja vehicle using lower front brace mounted front suspension

# Mathematical Modeling #2 - Front

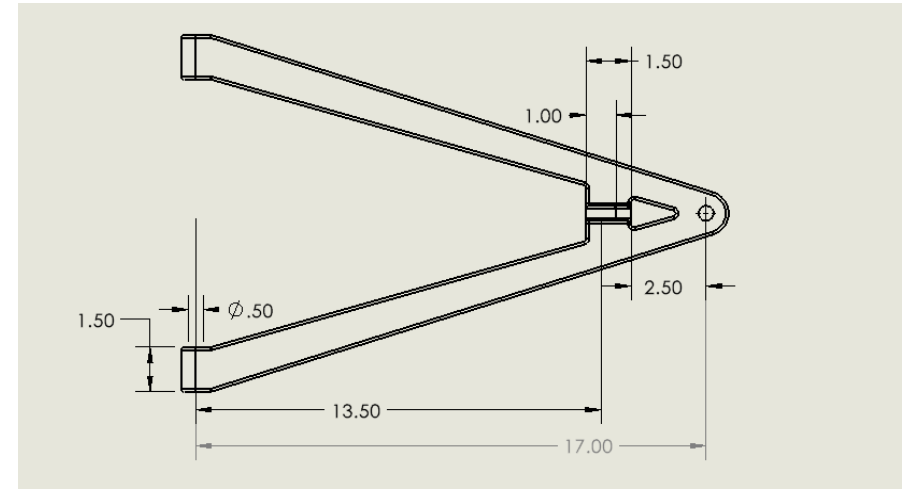
## A-Arm Design, Forces, and Inputs



Chassis Mount location

Shock Mount

Knuckle Mount



Dimensioned model of upper A Control arm.  
Dimensions are in Inches

# Mathematical Modeling #2 - Front

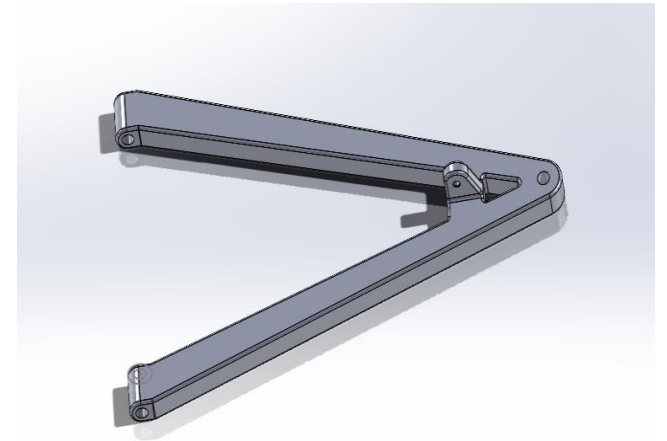
2-Dimensional shear force and bending moment diagrams on the upper A-arm during a max force compression

Instantaneous Shear force equation:

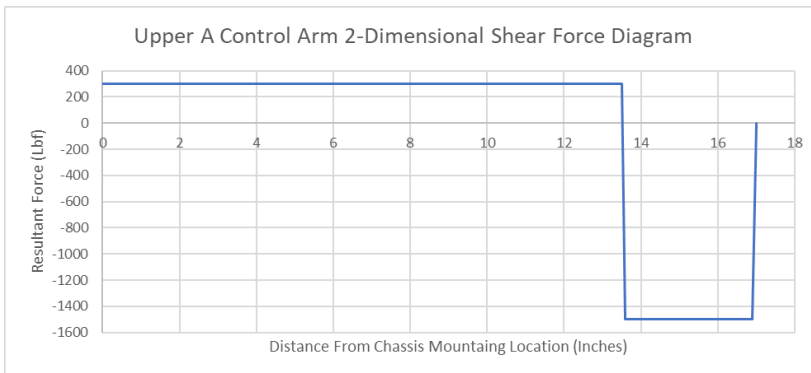
$$\tau = \frac{F}{A}$$

Bending Moment equation:

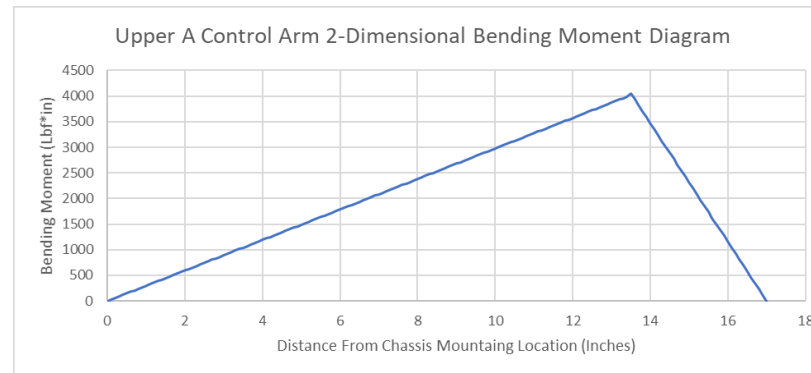
$$M = \int Sfd$$



SolidWorks Isometric view of upper A control arm with shock mounting uprights



Upper A Control Arm 2-Dimensional Shear Force Diagram in the Vertical Direction



Upper A Control Arm 2-Dimensional Bending Moment Diagram in the Vertical Direction

The upper A control arm must be designed to withstand a maximum bending moment of 4050 lbf\*in at 13.56 in from the chassis pivot point



# Mathematical Modeling #3 - Front

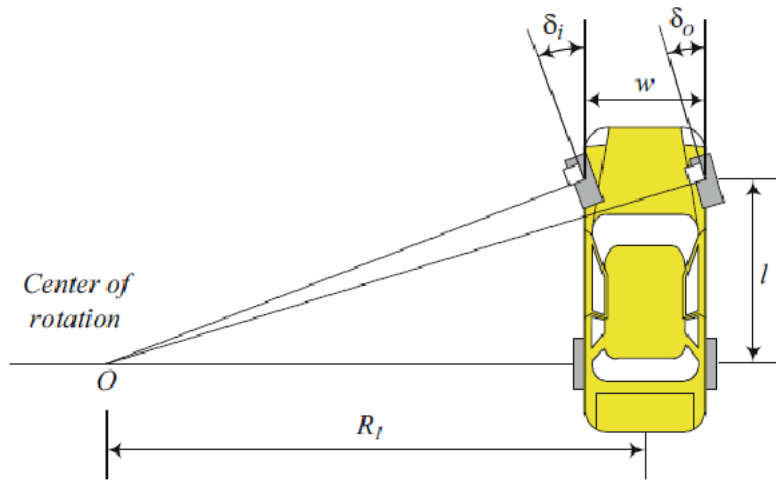
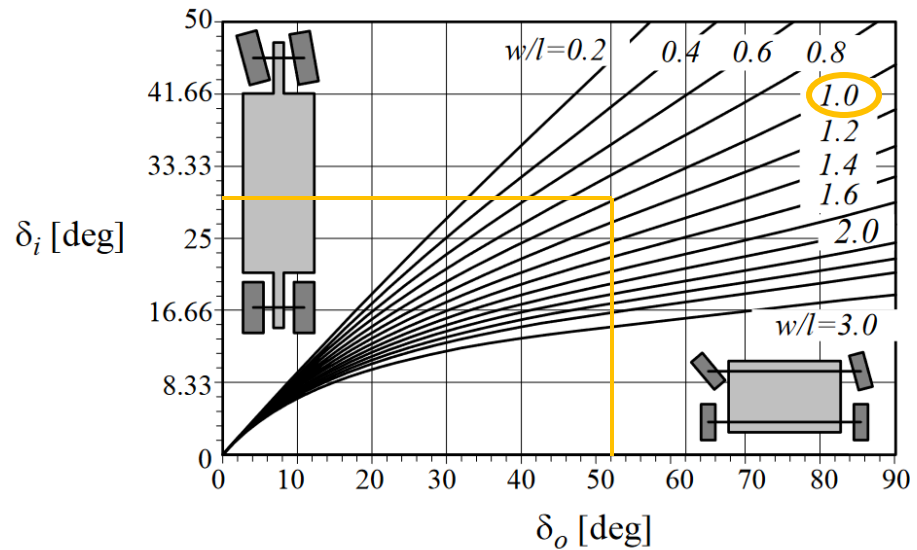


Diagram of Steering Radius correlated to Inner and Outer Steering Angles [24]

The above diagram and following equations used are under the assumption that the slip angles are close to or are  $0^\circ$  during a slow turn. To meet this condition, the chart on the right must be used to determine viable angles.



Effect of Width and Length on Viable Outer and Inner Steering Angles. These angles are calculated as a function of Lateral Force and Slip Angle [24]

# Mathematical Modeling #3 - Front

Calculating the average Steering Angle

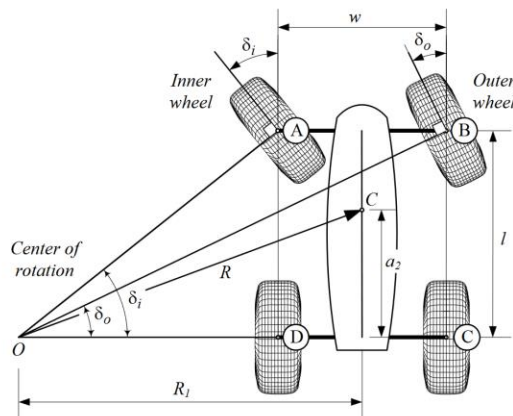
$$\cot \delta = (\cot \delta_o + \cot \delta_i) \frac{1}{2}$$

Steering Radius being calculated using center of Mass (a)

$$R = \sqrt{a_2^2 + l^2 \cot^2 \delta}$$

Calculating the Percent Ackerman Used

$$\%Ackermann = \frac{\delta_i - \delta_o}{\delta_i} * 100\%$$



Preliminary Measurements

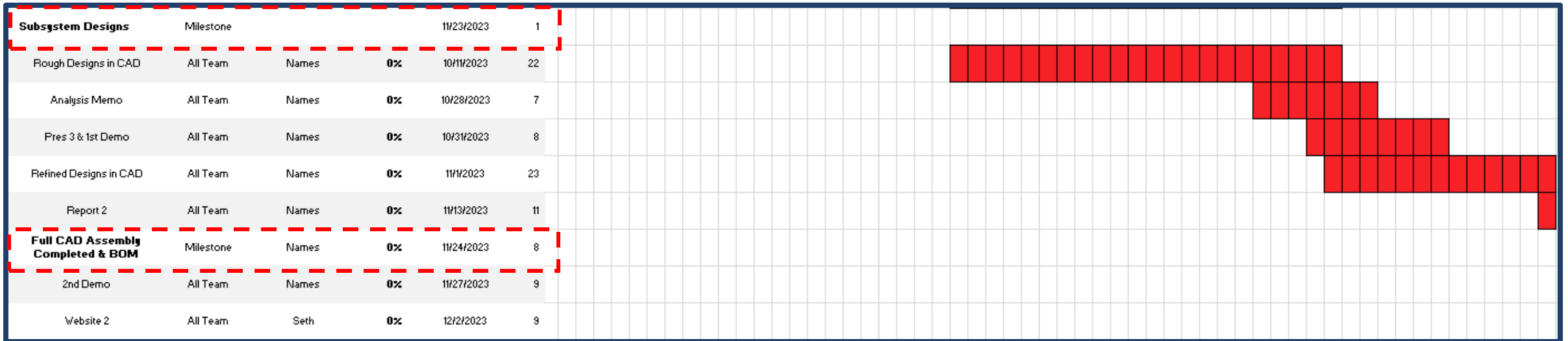
Wheel Center Length (l)	64in
Wheel Center Width (w)	64in
Inner wheel angle ( $\delta_i$ )	50°
Outer wheel angle ( $\delta_o$ )	30°
$\delta_{avg}$	40°
Rear wheel to center of gravity (a2)	32in

Results

Percent Ackerman Used	40%
Hypothetical Turning Radius (R)	6.89ft



# Project Schedule



**Progress:** On-Track...but need to be more aggressive!

**Next Steps (~1-1.5 Week(s) Out):** wireframe, front/rear hardpoints & geometry, initial powertrain calcs & geometry

# Project Budget

	Category	Relevant Items	Approximated Cost
1	Vehicle Expenses	Raw materials, hardware, engine & drivetrain components, tooling, harness & safety equipment, labor (if out-sourced manufacturing)	\$15,000
2	Spare Parts	Raw materials, labor (if out-sourced manufacturing), hardware	\$7,500
3	Competition Expenses	Registration, travel (hotel rooms, vehicle rentals, gas, etc.)	\$4,500
4	Contingency (5%)	Unpredicted Expenses	\$1,350
		<b>Total</b>	<b>\$28,350</b>

## Fundraising Plan

This is a steep budget that won't be covered by grants or singular donations...turn to sponsors!

### Potential Sponsors:

Copper State, Mother Road, Findley Toyota, Home Depot, ETM, Gore, HASS, N.A.P.A., etc.

### Sponsor Methodology:

Request funding in return for representation at competition via customized livery

**NAU**

# Rear Suspension Team

Joey Barta, Lars Jensen, Seth Deluca

# Benchmarking #1 – Rear Trailing Link

RIOT Racing



1st in suspension  
(4th overall)

LSU Baja Bengals



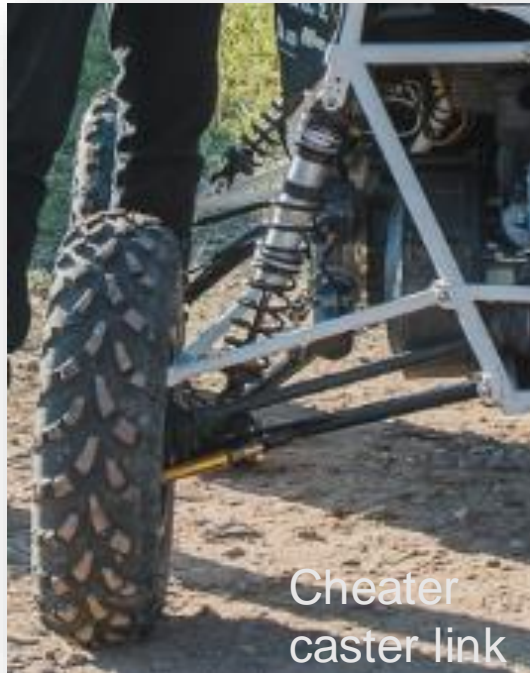
6th in suspension  
(21st overall)

Blue Jays Racing



43rd in suspension  
(30th overall)

# Benchmarking #2 – Rear Double A-Arm



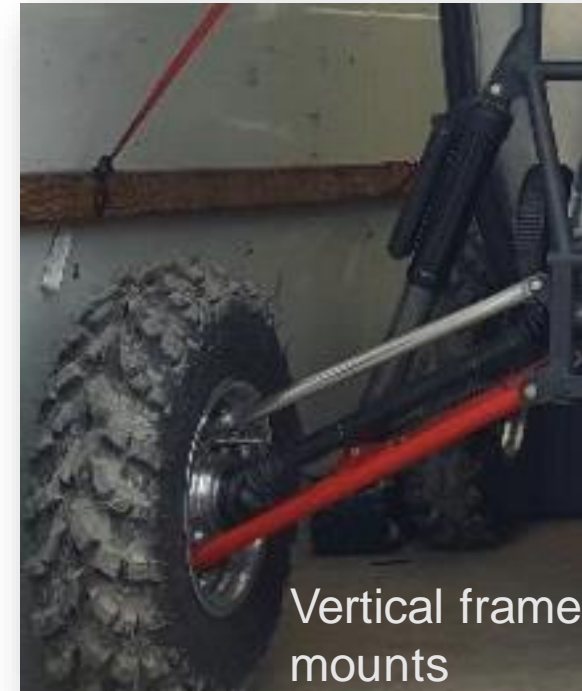
Cheater  
caster link

ETS – 3rd in  
suspension (5th  
overall)



Angled frame  
mounts

Oregon State University –  
3rd in maneuverability (2nd  
overall)



Vertical frame  
mounts

Cornell University – 2nd  
in maneuverability (3rd  
overall)



# Benchmarking #3 – Rear Single H-Arm



Northern Arizona University (2<sup>nd</sup> in suspension, 5<sup>th</sup> overall)



University of Michigan (4<sup>th</sup> overall)



Case Western Reserve University (6<sup>th</sup> overall)

# Customer & Engineering Requirements - Rear

## Customer Requirements

- Tunability
- Serviceability
- Reliability
- Ease of manufacturing
- Low cost
- Maximum Traction
- Maneuverability

## Engineering Requirements

- Decrease weight (lb.)
  - Rear suspension under 50 lbs.
- Increase strength (psi)
- Increase rearward axle path (in.)
  - 1 in. of rearward movement
- Increase linkage radii (in.)
  - 22 in. camber links
- Increase ground clearance(in)
  - 8 in. of ground clearance
- Vehicle width (in.)
  - Maximum vehicle width of 64 in.
- Decrease CV axle angle (degrees)
  - 180 degrees

# QFD Discussion - Rear

System QFD		Project: Lumberjack Motorsports SAE Baja Rear Suspension						
		Date: 9/19/23						
		Input areas are in yellow						
Decrease weight								
Increase strength	-3							
increase rearward axle path								
Increase linkage radii	-1							
Increase ground clearance	-6	3	1					
Vehicle width	1	3		6	2			
Decrease CV axle angle		3	-2	1				

Legend	
A	CWRUM
B	RIOT Baja
C	TS BAJA

		Technical Requirements							Customer Opinion Survey				
Customer Requirements	Customer Weights	Decrease weight	Increase strength	increase rearward axle path	Increase linkage radii	Increase ground clearance	Vehicle width	Decrease CV axle angle					
		1	2	3	4	5	6	7	Poor	Acceptable		Excellent	
		1	2	3	4	5	6	7	1	2	3	4	5
Tunability	2	2	3	8	7	3	2	7	C	A			B
Servicability	2	2	6						A		BC		
Reliability	5	3	9	3	2			7			A	B	C
Ease of manufacturing	3	6	7	1	1		1	1		C	A		B
Low cost	5	9	9	3	3		1	3			A	C	B
Maximum Traction	2	7		8	8		3	1				B	AC
Maneuverability	4	5	1	8	6	5	7	1				AB	C
Technical Requirement Units		lb	Psi	in	in	in	in	degrees					
Technical Requirement Units		<50	NA	<1	22	>8	<64	180					
Absolute Technical Importance		120	133	97	82	26	46	73					
Relative Technical Importance		2	1	3	4	7	6	5					

# Literature Review

-Joey Barta-

## Textbooks

- **W. F. Milliken and D. L. Milliken, Race Car Vehicle Dynamics [48]**
  - Highly regarded as the "bible" of suspension engineering. Authors developed many of the vehicle dynamics theories in the book
- **R. G. Budynas, Shigley's Mechanical Engineering Design [22]**
  - Useful source for failure prevention as well as design for mechanical elements
- **J. C. Dixon, Tires, Suspension and Handling [49]**
  - Detailed coverage of the theory and practice of vehicle cornering and handling

## Papers

- **Suspension Types – SUSPROG [50]**
  - illustrates potential rear suspensions with downloadable excel files pertaining to each.
- **J. Isaac-Lowry, "Suspension Design: Types of Suspensions," [51]**
  - short list of applicable designs to reference in the early stages of design
- **SLASIM: Suspension Analysis Program [52]**
  - software through MATLAB that analyzes functionality of suspension kinematics

## Online

- **Setup Suspension 101 [79]**
  - Article expands on preload, compression, rebound, ride-height, and crossover spacers tuning.
- **Suspension Geometry Calculator [65]**
  - Provides an intuitive, simple suspension geometry calculator

# Literature Review

-Seth DeLuca-

## Textbooks

- ***Vehicle Suspension System Technology and Design [60]***
  - *Chapter 4: Analysis and Design of Suspension Mechanisms*
- ***Geometric Design of Independent Suspension Linkages [61]***
  - Chapter 2.2: different mounting systems and joints

## Papers

- ***Fine-Tuning of the Suspension System [62]***
  - Includes information regarding things to look out for when tuning suspension
- ***Design Analysis of 3 Link Trailing Arm [63]***
  - In depth to 3-Link Trailing Arm
- ***Design Analysis of H-arm with Camber Link [64]***
  - Looks into H-arm Suspension

## Online

- ***Racing Aspirations suspension Geometry [65]***
  - Quick suspension software analysis
- ***Spring rate and wheel rate calculator [66]***
  - Calculates shock angle when given different parameters based on a simple geometry

# Literature Review

-Lars Jensen-

## Textbooks

- **Performance Vehicle Dynamics: Engineering and Applications [67]**
  - Chapter 7 – Suspension Kinematics, Chapter 8 – Dynamic Modelling of Vehicle Suspension
- **The Multibody Systems Approach to Vehicle Dynamics [68]**
  - Chapter 4 – Modelling and Analysis of Suspensions Systems

## Papers

- **Suspension Design and testing of an All-Terrain Vehicle using Multi-body dynamics Approach [69]**
  - Flow of design calculations for suspension parameters
- **Optimal Design of Suspension System of Four-wheel Drive Baja Racing [70]**
  - Geometric design of rear suspension
- **Design and Optimization of Rear Wheel Assembly for All-Terrain Vehicle [71]**
  - FEA analysis of rear knuckle and hub

## Online

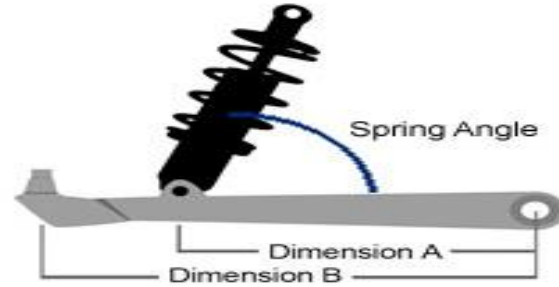
- **Float 3 EVOL RC2 Factory Series Owner's Manual [72]**
  - Shock service and tuning
- **A Square C & D “BAJA ATV Videos” Playlist [73]**
  - SolidWorks modeling of suspension systems and knuckles

# Mathematical Modeling #1 - Rear

What angle should the shock be to the suspension system?

## Assumptions:

Corner weight  $\cong$  150 lbs  
 Unsprung weight  $\cong$  45 lbs  
 Dimension A  $\cong$  16 in  
 Dimension B  $\cong$  20 in  
 Spring angle  $\cong$  80 degrees  
 Shock ride height = 2.27 in



Spring angle  $\cong$   
 60,70,80,90 degrees

Formulas [66]

*Sprung height*  
 = *Corner weight*  
 - *Unsprung weight*

*Motion ratio* =  
 $\frac{\text{Dimension A}}{\text{Dimension B}}$   
 \*  $\sin(\text{Spring angle})$

*Static load*  
*Sprung weight*  
 =  $\frac{\text{Static load}}{\text{Motion ratio}}$

*Spring rate*  
*Static load*  
 =  $\frac{\text{Spring rate}}{\text{Shock ride height}}$

*Effective wheel rate*  
 = *Spring rate*  
 \* *Motion ratio*<sup>2</sup>

*Spring rate*  
 =  $\frac{(\text{Corner weight} - \text{Unsprung weight})}{\frac{\text{Dimension A}}{\text{Dimension B}} * \sin(\text{Spring angle})} \div \text{Shock ride height}$

*90 degrees*

*Spring rate* = 57.76 lb / in  
*Effective wheel rate* = 36.96 lb / in

*80 degrees*

*Spring rate* = 58.65 lb / in  
*Effective wheel rate* = 36.40 lb / in

*70 degrees*

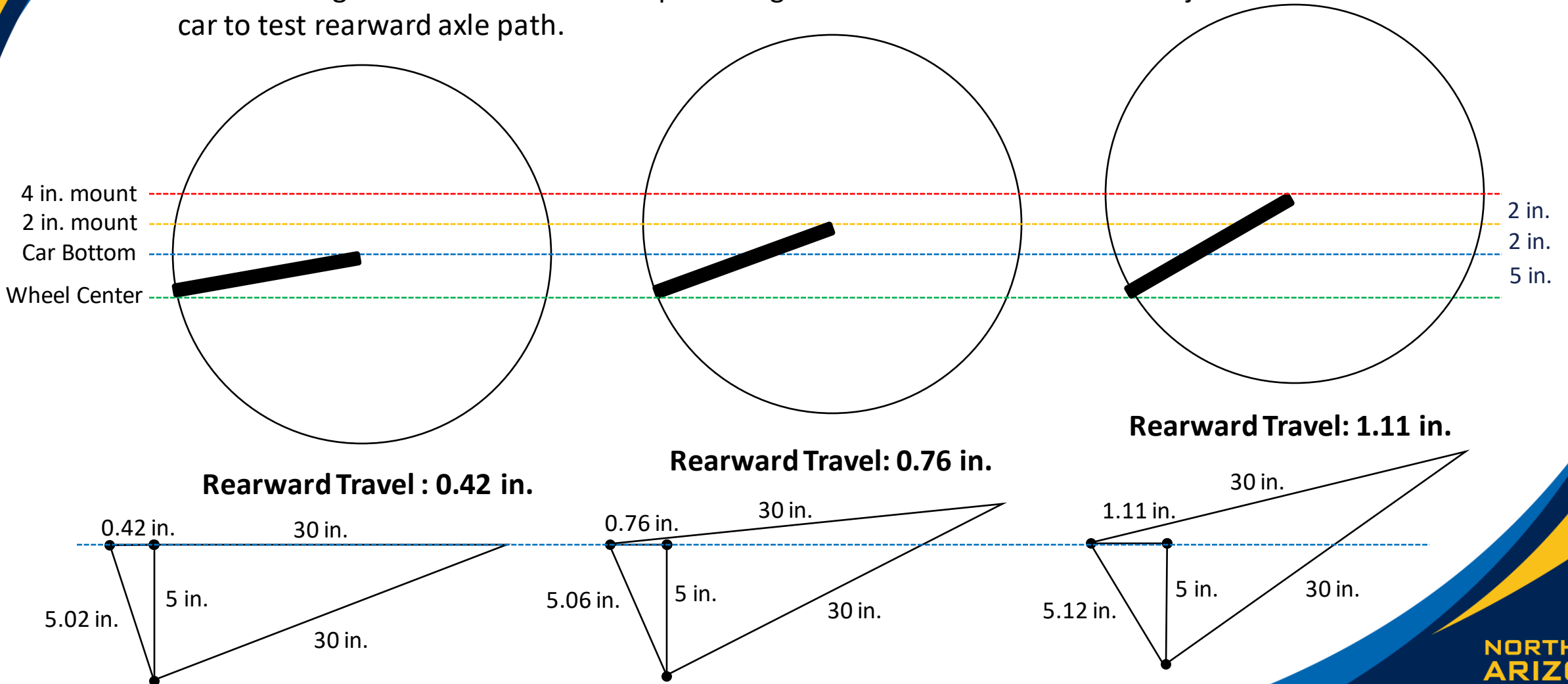
*Spring rate* = 61.46 lb / in  
*Effective wheel rate* = 34.73 lb / in

*60 degrees*

*Spring rate* = 66.69  
*Effective wheel rate* = 32.01

# Mathematical Modeling #2 - Rear

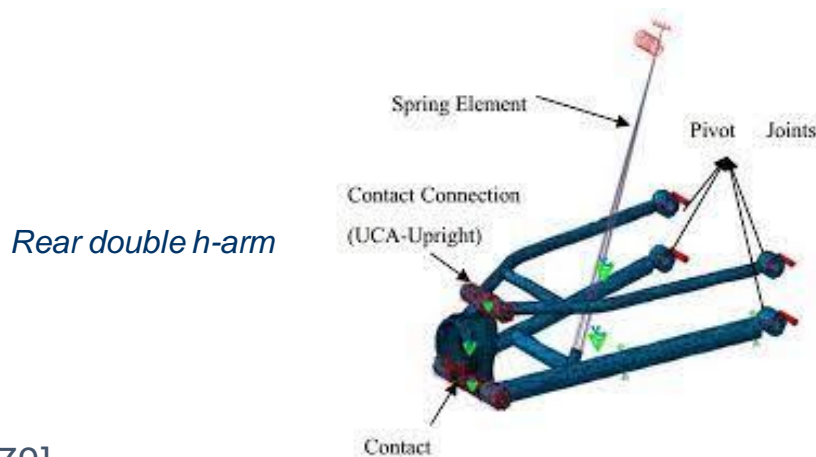
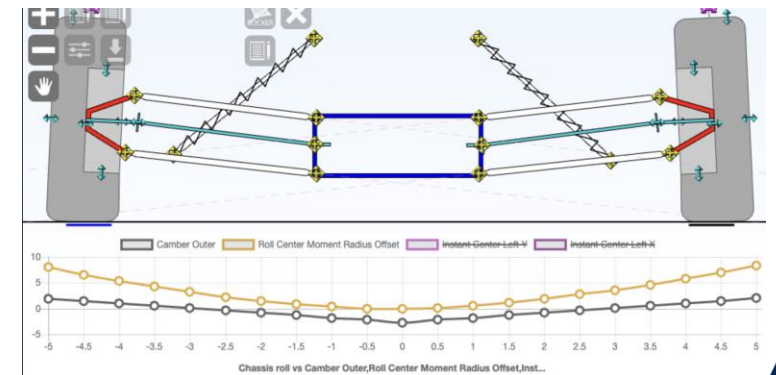
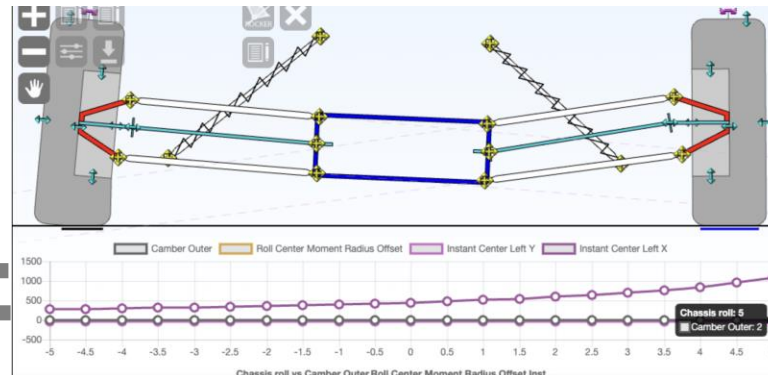
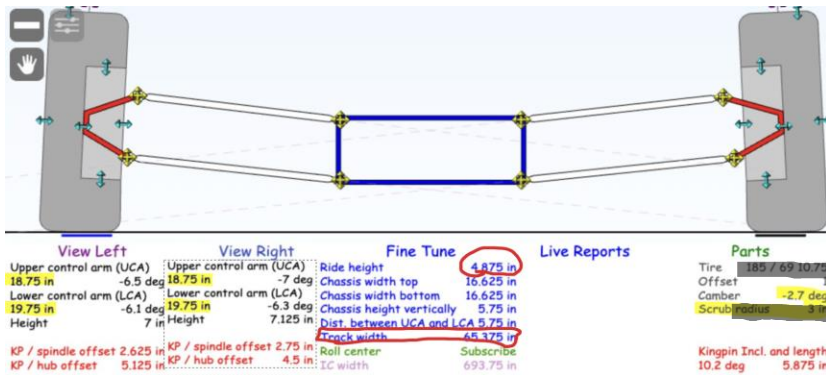
30 in. trailing link with different front pivot heights relative to the bottom of baja car to test rearward axle path.





# Mathematical Modeling #3 - Rear

How does change in upper a-arm position/length affect camber gain?

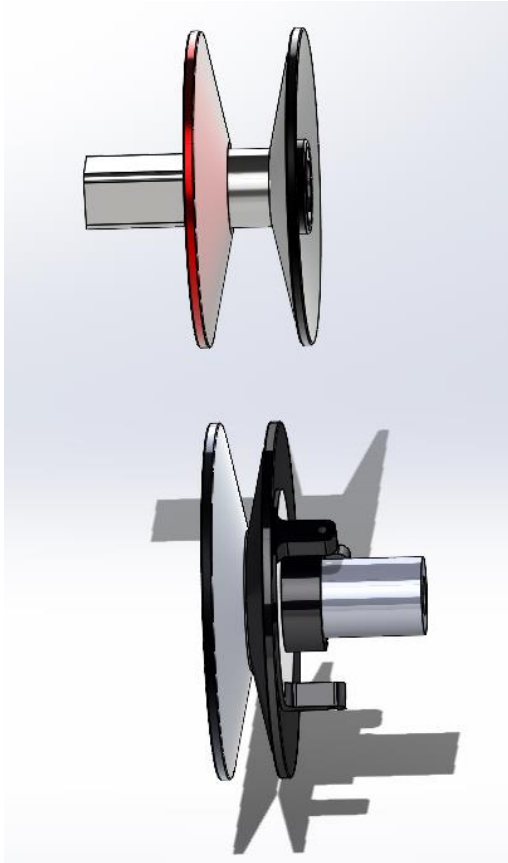


Goal: design a rear suspension with 90° camber at unsprung weight and negative camber (around 3-6°) under hard cornering.

# Drivetrain Team

Henry Van Zuyle, Donovan Parker, Ryan Fitzpatrick,  
Jarett Berger

# Benchmarking #1 – Drive



*New CVT Sheave Geometry with Appropriate Total Range*

## Total gear ratio change

Researched other transmissions to find out what is a good total transmission ratio change

- Michigan 4.0:1 to .75:1 total range 5.333
- Cornell 2.23:1 to .5:1 total range 4.46
- Gaged 3.9:1 to .9:1 total range 4.333
- New ECVT 2.66:1 to .588:1 total range 4.53

# Benchmarking #2 - Drive



*Purdue University*

## Rear End Optimization Integrated CV Cups



*Cornell  
University*



*Northern  
Arizona  
University*

# Benchmarking #3 - Drive



CV joint

U-joint

## Front End Optimization

Use CV joint rather than U-joint

- No binding with CV joint
- Increased range of angle
- Load is equally distributed

*CV joint and U-joint Comparison*



*ETS Baja CV Axles*



*Michigan Baja Front End*

# Customer & Engineering Requirements - Drive

## Customer Requirements

- High top speed
- Maximum efficiency
- High torque
- High service life
- Low weight
- High transmission range

## Engineering Requirements

- 40mph top speed
- 80% drivetrain efficiency
- 400lb-ft of torque to the wheels
- 1000-hour service life
- Total drivetrain weight (w/out engine) 60lbs
- 1:4.5 total transmission range

# QFD Discussion - Drive

input areas are in yellow

1	top speed																			
2	drivetrain efficiency																			
3	torque to the wheels	6																		
4	service life																			
5	total system weight (w/out engine)		3																	
6	total transmission range	6																		
7	Meets HROE Guard specifications																			

		Technical Requirements								Customer Opinion Survey				
		top speed	drivetrain efficiency	torque to the wheels	service life	total system weight (w/out engine)	total transmission range	Meets HROE Guard specifications	1 Poor	2	3 Acceptable	4	5 Good	
	<b>Customer Needs</b>													
1	fast	5	9	6	6	3	6	6	1	C		B	A	
2	High efficiency	3	9	9	6	6	3	6	3	C		B	A	
3	fast acceleration	5	3	6	9	3	9	9	1	C		B	A	
4	durable	1	1	6	1	9	1	4	9		AC		B	
5	can crawl and go fast	4	9	6	9	3	3	9	1	C		B	A	
6	NEEDS TO BE SAFE	5	1	1	1	6	1	1	9		C	B	A	
7	Aesthetically Pleasing	3	9	1	9	3	9	9	3		C	A	B	
8														
9														
10														
11														
	<b>Technical Requirement Units</b>	MPH	Unitless	Lb/FT	hours	lbs	Unitless	N/A						
	<b>Technical Requirement Targets</b>	4 156 40	6 125 80	3 162 400	7 108 1000	5 129 60	2 165 01,04,5	N/A						
	<b>Absolute Technical Importance</b>	4 156 40	6 125 80	3 162 400	7 108 1000	5 129 60	2 165 01,04,5	1 86						
	<b>Relative Technical Importance</b>	4 156 40	6 125 80	3 162 400	7 108 1000	5 129 60	2 165 01,04,5	1 86						

Drivetrain QFD

# Literature Review

-Henry Van Zuyle-

## Textbooks

- **Shigley's Mechanical Engineering Design [22]**
  - Chapter 17, Flexible Mechanical Elements
- **Machinery's Handbook [25]**
  - Chapter, Gearing

## Papers

- **US Patent US20180172150A1, Electromechanically actuated continuously variable transmission system and method of controlling thereof [37]**
  - ETS ECVT patent
- **An Experimentally-Validated V-Belt Model for Axial Force and Efficiency in a Continuously Variable Transmission [38]**
  - Factors that effect CVT efficiency
- **Modeling and Tuning of CVT Systems for SAE® Baja Vehicles [40]**

## Online

- **Shaft Splines & Serrations [42]**
  - Spline strength and geometry
- **Altair Motion View: CVT Model [43]**
  - Helped me develop my CVT design software



# Literature Review

-Ryan Fitzpatrick-

## Textbooks

- **Shigley's Mechanical Engineering Design [22]**
  - Chapters 13 & 14, Gears – General & Spur and Helical Gears
- **Machinery's Handbook [25]**
  - Chapter 12, Gearing

## Papers

- **Methodology for Designing a Gearbox and its Analysis – IJERT [56]**
  - Gearbox Design General
- **Design and Analysis of Gearbox for SAE Baja Competition – IJERT [55]**
  - Gearbox Design for SAE
- **Lightweight Design of Gearbox Housing of Baja Racing Car Based on Topology Optimization – Journal of Physics [81]**
  - Gearbox Housing Design for optimization

## Online

- **Gear Design by AGMA Theory – The Engineering Blog [82]**
  - AGMA theory source that includes lube factor
- **A Look at Belt, Chain and Gear Drive Technology – Power Transmission Engineering [83]**
  - Power Transmission Options Discussion
- **Chain Sprocket Calculator [84]**
  - Used to calculate the chain drive options

# Literature Review

-Donovan Parker

## Textbooks

- **Shigley's Mechanical Engineering Design [22]**
  - Chapter 17, Flexible Mechanical Elements
- **Machinery's Handbook [25]**
  - Machine Elements -> Flexible Belts and Sheaves/ Transmission Chains

## Papers

- **Design Analysis and Fabrication of the Powertrain System for All-Terrain Vehicle – IJERT [80]**
  - Calculations
- **SAE Baja '24 Rule Book [89]**
  - Belt, Gear, and Chain Drives
- **Design of a Drivetrain for SAE Baja Racing Off-Road Vehicle – IJAEMS [85]**
  - Powertrain

## Online

- **Belts and Chains Play to Their Strengths – Power Transmission Engineering [88]**
- **Belts/ Other Drives – Baja SAE Forums [87]**
- **Belt and Chain CVT: Dynamics and Control – Mechanism and Machine Theory [86]**

# Literature Review

-Jarett Berger

## Textbooks

- **Shigley's Mechanical Engineering Design [22]**
  - Chapter 14, Spur and Helical Gears, Chapter 18, Power Transmission
- **Machinery's Handbook [25]**
  - Gears, Splines, and Cams

## Papers

- **Spur Gear Designing and Weight Optimization – IJERT [45]**
  - Designing spur gear
- **Design, Analysis, and Simulation of a Four-Wheel-Drive Transmission for an All-Terrain Vehicle – SAE [54]**
  - 4WD analysis
- **Design and Analysis of Gearbox for SAE Baja Competition – IJERT [55]**
  - Gearbox analysis

## Online

- **Methodology for Designing a Gearbox and its Analysis - IJERT [56]**
  - Design approach for gearbox
- **Design and Analysis of Gearbox with Integrated CV Joints – Blog [57]**
  - CV joints integrated with output shaft

# Mathematical Modeling #1 - Drive

$$\Phi_d = \pi - 2\sin^{-1} \frac{D-d}{2C} = 2.282\text{rads} = 130.757^\circ$$

$$\Phi_D = \pi + 2\sin^{-1} \frac{D-d}{2C} = 4.001\text{rads} = 229.243^\circ$$

$$L = \sqrt{4C^2 - (D-d)^2} + \frac{1}{2}(D\Phi_D + d\Phi_d) = 110.92 \text{ in.} = 9.24 \text{ ft}$$

$$F_1 = \frac{T \times 12}{r} = 1048 \text{ lbf (Tension Side Force)}$$

$$F_2 = F_1 - \frac{2T}{d} = 0 \text{ lbf (Slack Side Force)}$$

Where:

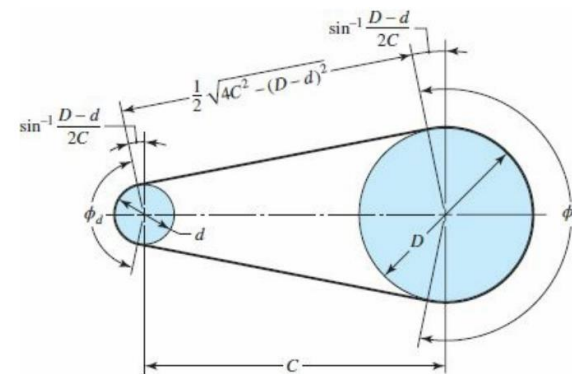
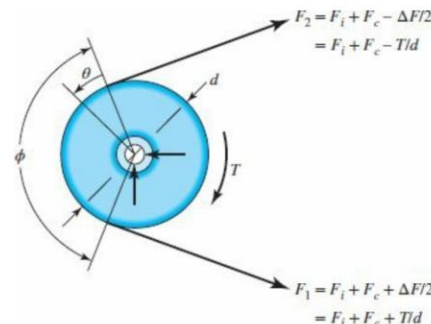
$\Phi_d$  = Smaller pulley wrap angle

$\Phi_D$  = Larger pulley wrap angle

L = Belt length

$F_1$  = Tension side force

$F_2$  = Slack side force



## Wrap angle and Centrifugal Force

Creep prevention and Maximum power output and efficiency for 4WD integration

- Wrap angle calculation
- Belt Length calculation
- Centrifugal force calculation

# Mathematical Modeling #2 - Drive

If  $\beta \geq \pi$

$$T_0(lbf) = \frac{2 \sin\left(\frac{\beta}{2}\right) * \left[2F_{Clamp} \tan\left(\frac{\phi}{2}\right) + \frac{1}{12} M_{Belt} * R^2 * \omega^2\right]}{\cos\left(\frac{1}{2}(\beta - \pi)\right) * (e^{\mu_e \beta} + 1)}$$

If  $\beta \leq \pi$

$$T_0(lbf) = \frac{2 \sin\left(\frac{\beta}{2}\right) * \left[2F_{Clamp} \tan\left(\frac{\phi}{2}\right) + \frac{1}{12} M_{Belt} * R^2 * \omega^2\right]}{\cos\left(\frac{1}{2}(\pi - \beta)\right) * (e^{\mu_e \beta} + 1)}$$

$$\tau_{MaxSecondary}(ft.*lbf) = (T_{Taught\_Secondary} - T_{Slack\_Secondary}) * \frac{Radius_{Secondary}}{12}$$

$$T_1 = T_0 e^{\mu_e \beta}$$

$$T_{1\_Primary}(lbf) = T_{1\_Secondary}$$

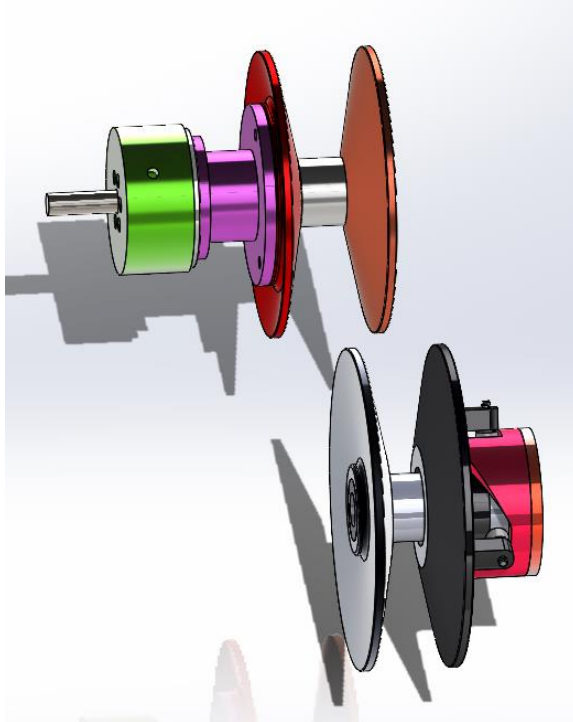
Governing Equations for CVT Geometry and Forces

## CVT Geometry and actuation force requirements

Utilized a matlab script to iterate through

- CVT sheave geometry to ensure large enough total ratio change
- Secondary cam geometry to generate sufficient clamping force

# Mathematical Modeling #2 - Drive

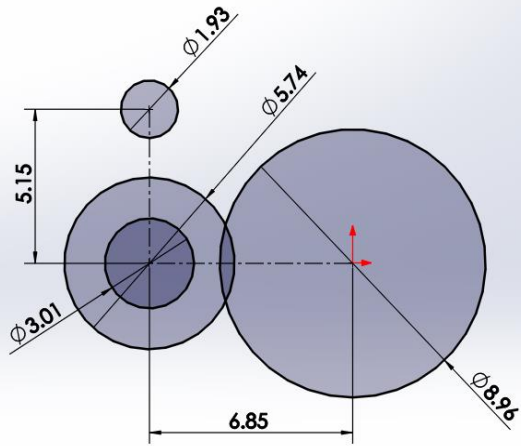


*CVT design with non-optimized components*

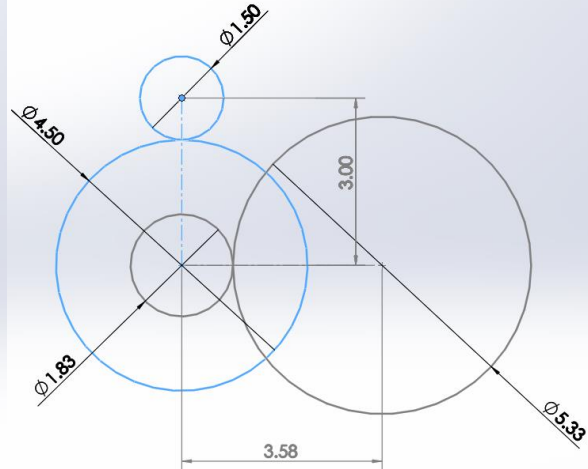
## Results

- Center to center is 9.5 inches
- Sheave angle is 12.77 degrees
- Sheave diameters clear each other and the motor
- CVT has a competitive range of 4.533
- Max actuation force from primary is 301 LbF

# Mathematical Modeling #3 - Drive



*Chain Drive Basic Sizing*



*Gearbox Basic Sizing*

By taking the torques and angular velocity on each stage the minimum ANSI Chain Number of #50 was determined. These values were input into an online sprocket calculator [84] and the above dimensions were determined for a chain drive.

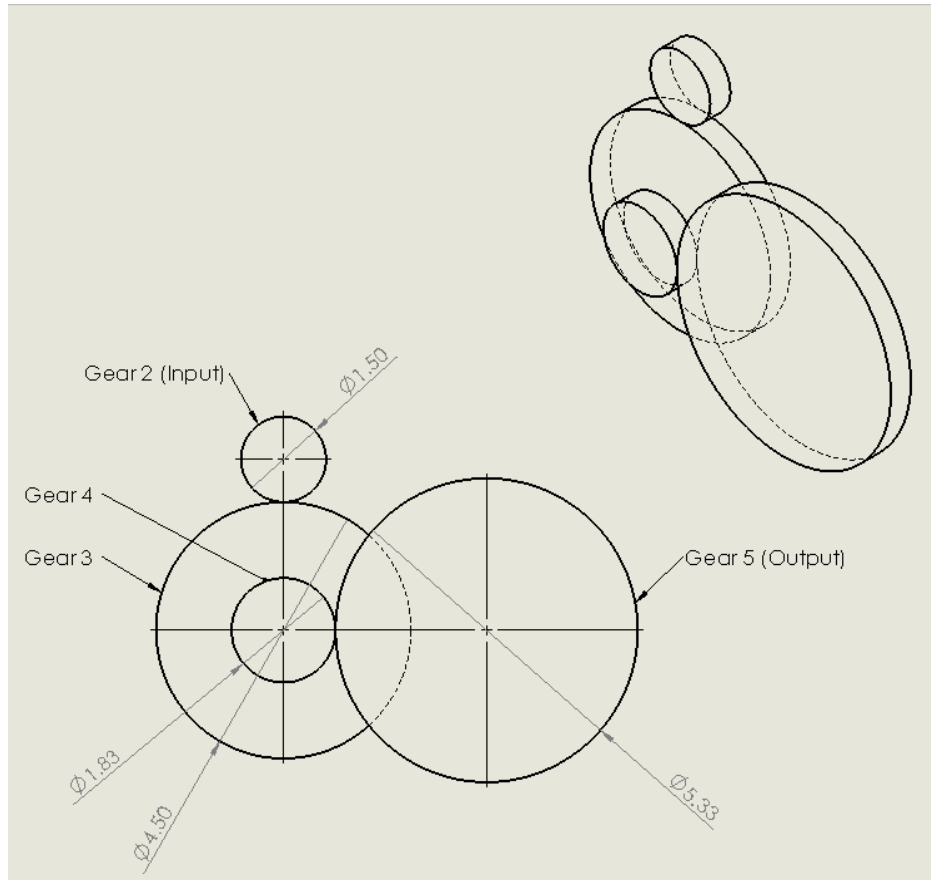
## Rear Power Transmission

Did research and basic calculations to determine if power should be transmitted via chain or gear drive.

- Gearbox is slightly more efficient (Chain drives have efficiencies “up to about 98%” [83] versus gearboxes efficiencies of “less than 2%” [22]).
- Gearbox takes up 60% less space (see modeling to the left)
- Gearbox requires less maintenance [83]
- The top teams run gearboxes

From these calculations, we have decided to move forward with a gearbox as opposed to a chain drive.

# Mathematical Modeling #4 - Drive



Preliminary Gearbox Design

## Rear Gearbox Design

Looking to achieve a reduction of ~9.2 via a two-stage compound gear train while minimizing the space occupied by the gearbox to allow for better rear suspension geometry.

$$\text{Diametral Pitch : } P = \frac{N}{d} ; P \text{ (1st Stage)} = 16 \frac{\text{teeth}}{\text{in}} ; P \text{ (2nd Stage)} = 12 \frac{\text{teeth}}{\text{in}}$$

$$\text{Train Value : } e = \frac{\text{product of driving tooth numbers}}{\text{product of driven tooth numbers}} ; e_{\text{target}} = \frac{1}{9.2}$$

$$\text{Minimum Pinion Teeth : } N_{P,\text{min}} = \frac{2k}{(1+2m)\sin^2(\Phi)} (m + \sqrt{m^2 + (1+2m)\sin^2(\Phi)})$$

where... k = 1 (for full-depth teeth)

$\Phi = 20\text{deg}$  (standard pressure angle)

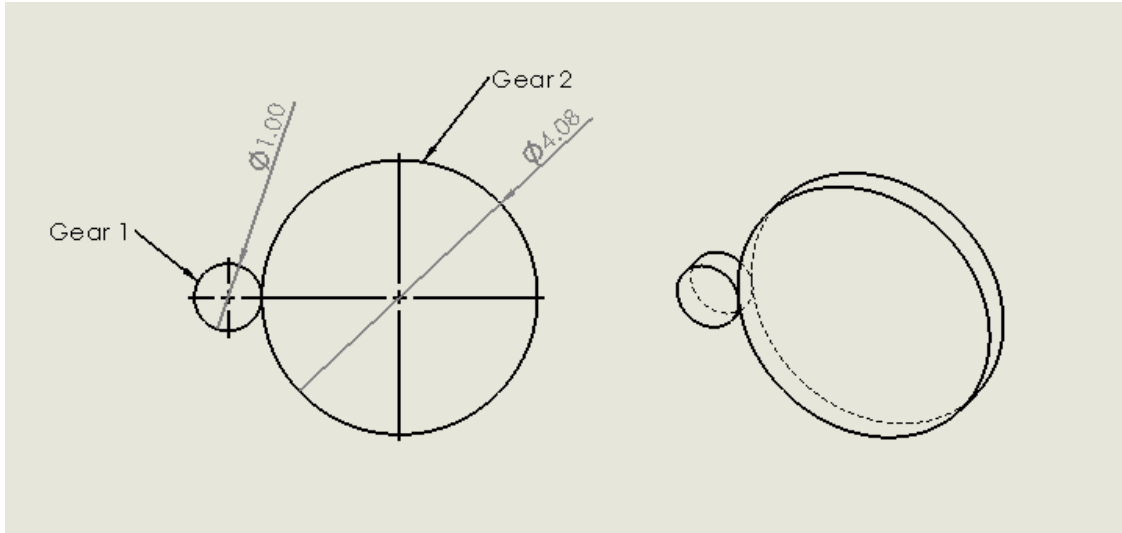
Input RPM = 1200

Output RPM = 120-140

Gear	d (diameter, in)	N (number of teeth)
2	1.5	24
3	4.5	72
4	1.83	22
5	5.33	64



# Mathematical Modeling #5 - Drive



Preliminary Gear Design

	Gear 1	Gear 2
Teeth	12	49
Pitch Dia. (in)	1	4.08333
Dia Pitch (teeth/in)	12	12

Gear Calculations

## Front Gearbox Design

- 1 stage gear reduction to achieve more torque and steering traction
- Size of gear teeth will minimize front end space so that suspension and steering designs are optimized

Table 13-2 Tooth Sizes in General Uses

Diametral Pitch $P$ (teeth/in)	
Coarse	2, $2\frac{1}{4}$ , $2\frac{1}{2}$ , 3, 4, 6, 8, 10, <span style="border: 1px solid black; padding: 2px;">12</span> , 16

$$\text{Minimum Teeth for Pinion: } N_p = \frac{2k}{3\sin^2\phi} (1 + \sqrt{3\sin^2\phi})$$

$$\text{Maximum Teeth for Gear: } N_G = \frac{N_p \sin^2(\phi) - 4k^2}{4k - 2N_p \sin^2(\phi)}$$

$K$  (Full-Depth) = 1

Pressure angle  $\Phi = 20\text{deg}$

# Frame Team

Cooper Williams, Gabriel Rabanal, Antonio Sagaral

# Benchmarking – Frame



ETS  
Rear Brace  
1st overall in Oregon 2023



University of Michigan  
Rear Brace  
4th overall in Oregon 2023



Cornell University  
Front Brace  
4th overall in NY 2020

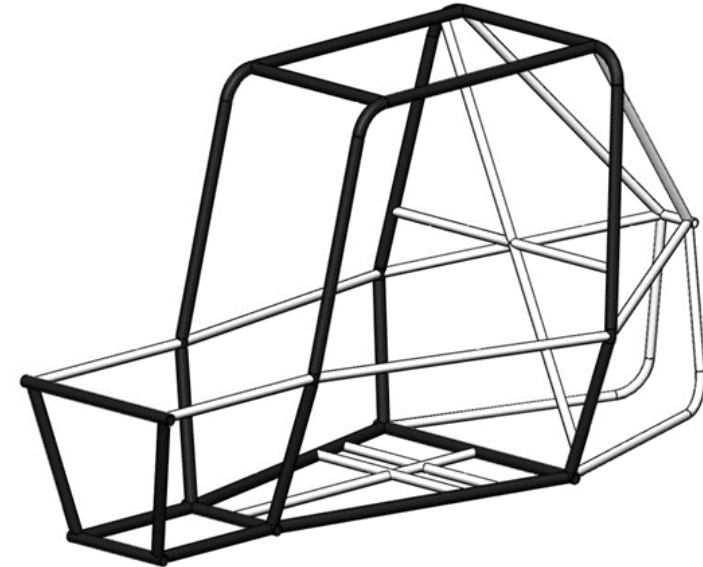
# Customer & Engineering Requirements - Frame

## Customer Requirements

- Lightweight
- Durable
- Easy to manufacture
- Satisfy SAE Baja frame guidelines
- Affordable
- Maneuverable
- Rigid
- Fast

## Engineering Requirements

- Steel carbon content greater than 18%
- RRH angle between 0 and 20 deg
- Primary members minimum specs: 0.984in O.D. and 0.062in wall thickness
- Secondary members minimum specs: 1.0in O.D. and 0.035in wall thickness



Roll Cage, Primary Members (filled in black), Rear Braced Frame

# QFD Discussion - Frame

System QFD

Project:	Baja 24 Frame
Date:	9/14/23

Legend  
 A [Allied/University](#)  
 B [University](#)  
 C [University](#)

		Technical Requirements						Customer Opinion Survey					
		Decrease weight	Decrease length of body	Decrease width of body	Decrease Cost	Increase aerodynamics	Increase strength of frame	1 Poor	2	3 Acceptable	4	5 Excellent	
2	Decrease weight	6											
3	Decrease length of body	3	-9										
4	Decrease width of body	-9	3	3									
5	Decrease Cost	-3	-6	3	-3								
6	Increase aerodynamics	6	-3	-9	-6								
7	Increase strength of frame												
Customer Needs	Customer Weights	Decrease weight	Decrease length of body	Decrease width of body	Decrease Cost	Increase aerodynamics	Increase strength of frame	1	2	3	4	5	
1	Rigid	3	1	6	3	3	1	9				ABC	
2	Easy to manufacture	3	3	3	1	3	6	3		B	AC		
3	Maneuverable	2	3	9	9	1	1	3				ABC	
4	Aesthetics	1	3	1	3	3	9	1		C	B	A	
5	Durable	2	3	1	3	3	3	9		AC	B		
6	Satisfy SAE Baja Frame Guidelines	4	3	1	6	3	1	6				ABC	
7	Stable	3	1	3	9	1	3	6				C AB	
8	Fast	3	6	3	3	9	9	3				BC A	
9	Lightweight	4	9	6	3	9	3	6				ABC	
10	Affordable	3	9	6	3	9	6	6				ABC	
Technical Requirement Units		lbs	in	in	\$	lbf	psi						
Technical Requirement Targets													
Absolute Technical Importance		3	123	5	112	64	4	120	64	2	134	6	108
Relative Technical Importance		3	123	5	112	64	4	120	64	2	134	6	108

## Main Customer Needs

- Satisfies SAE Rules
- Lightweight
- Affordable
- Easy to manufacture

## Main Engineering Requirements

- Increased frame strength
- Decrease cost
- Decrease weight
- Decrease body width

# Literature Review

-Gabriel Rabanal-

## Textbooks

- ***Materials selection in mechanical design* [18]**
  - Ch 5-6: material selection process
- ***The Automotive Chassis: Engineering Principles* [14]**
  - Ch 6: loading effects on chassis types, braking behaviors

## Papers

- ***A novel approach for design and analysis of an all-terrain vehicle roll cage* [15]**
  - Impact analysis for rear braced frame design
- ***Computational analysis for improved design of an SAE Baja frame structure* [16]**
  - Rear braced frame analysis and materials discussion
- ***Design and FE analysis of chassis for solar powered vehicle* [17]**
  - Frame material comparison

## Online

- ***Mini Baja Vehicle Design Optimization* [19]**
  - Bracing analysis/failure testing
- ***SolidWorks BAJA SAE Tutorials - How to Model a Frame (Revised)* [20]**
  - CAD Tutorial, Design of SAE Baja frame in SolidWorks

# Literature Review

-Antonio Sagaral-

## Textbooks

- ***Shigley's Mechanical Engineering Design [22]***
  - Ch 2-22: Materials selection
- ***Fundamentals of Machine Component Design [29]***
  - Ch 11: Rivets, Welding, and Bonding

## Papers

- ***Design, Analysis and Optimization of a BAJA-SAE Frame [39]***
  - FEA Analysis and Material options
- ***Design and Construction of a Space-frame Chassis [41]***
  - Suspension forces and FEA Analysis
- ***DESIGN AND STRUCTURAL ANALYSIS OF BAJA FRAME WITH CONVENTIONAL AND COMPOSITE MATERIALS [44]***
  - Impact analysis and stress calculations

## Online

- ***[Front Impact Test & Meshing] BAJA SAE Roll Cage/Frame Design in ANSYS Workbench Static Structural [53]***
  - Impact testing points and strategies
- ***Baja SAE Frame Investigations [47]***
  - Investigation of rear vs front brace frames

# Literature Review

-Cooper Williams

## Textbooks

- ***Shigley's Mechanical Engineering Design* [22]**
  - Chapter 9: Welding, Bonding, and the Design of Permanent Joints
- ***The Automotive Chassis: Engineering Principles* [14]**
  - Ch 6.1: Vehicle and body center of gravity

## Papers

- ***Design and Analysis of Chassis for SAE BAJA Vehicle* [19]**
  - Member Stress Analysis
- ***Mathematical Model for Prediction and Optimization of Weld Bead Geometry in All-Position Automatic Welding of Pipes* [25]**
  - Strength of Welds given several factors
- ***Design, analysis and optimization of all-terrain vehicle chassis ensuring structural rigidity* [21]**
  - Provides analysis for different frame impacts

## Online

- ***Design Judging Discussion* [28]**
  - Discusses how some judges approach scoring vehicles
- ***Getting Started with weldments in SOLIDWORKS* [27]**
  - Introduction to Weldments, Manufacturing



# Mathematical Modeling #1-Frame

## Densities of Suitable Steels

Material	Density (lb/in <sup>3</sup> )
1018 LC, CD Steel	0.284
1020 LC, CD Steel	0.284
4130 MC, CD Steel	0.284

Table: Density Comparison

## Material/Cost Comparison

- *Each year, about 80 feet is purchased for frame manufacturing*
- *Only use about 90% of the 80ft*

$$\rho = \frac{m}{V}$$

$\rho$ : density, lb/in<sup>3</sup>

$m$ : mass, lbm

$V$ : volume, in<sup>3</sup>

# Mathematical Modeling #2 - Frame

Steel Type (AISI)	1018	4130	1020	1020
O.D. (in)	1	1	1	1.125
Wall thickness (in)	.12	.12	.12	.12
$S_b$ (klb*in)	3.513	4.128	3.323	4.381
$A_s$ (in <sup>2</sup> )	.3317	.3317	.3317	.3789

## Equations

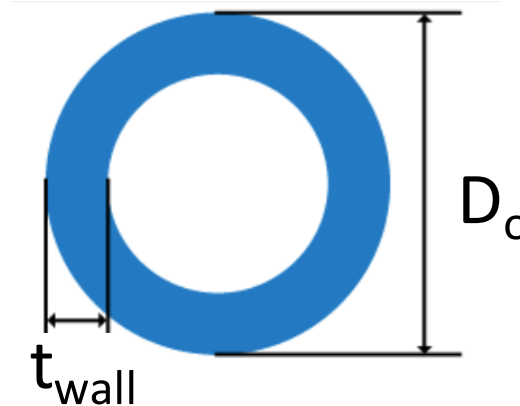
- $S_b = \frac{S_y I}{c}$
- $A_s = \pi \left(\frac{D_o}{2}\right)^2 - \pi \left(\frac{D_o}{2} - t_{wall}\right)^2$

## Frame Material Comparison

Calculations performed for primary members

SAE Specifications:

- Minimum OD of 1"
- Standard minimum thickness of .12"



# Mathematical Modeling #3 - Frame

## Primary Member Tube Specs

### Equations

- $S_b = \frac{S_y I}{c}$
- $K_b = EI$

	Base 1018 requirements	4130	4130	4130
O.D. (in)	0.984	1	1.125	1.25
I.D. (in)	0.748	0.76	0.959	1.12
Wall thickness (in)	0.118	0.12	0.083	0.065
$K_b$ (klb*in)	<b>948.6</b>	<b>948.6</b>	<b>1076.2</b>	<b>1235.5</b>
$S_b$ (klb*in)	<b>3.513</b>	<b>4.128</b>	<b>4.163</b>	<b>4.301</b>

# Thank you!



# Bibliography

- [1] “Nau Sae Baja team on Instagram: ‘so excited to see where our team members end up in the next few years...and that’s a wrap!’” Instagram, [https://www.instagram.com/p/COd0nAAL000/?utm\\_source=ig\\_web\\_copy\\_link&igshid=MzRIODBiNWFIZA](https://www.instagram.com/p/COd0nAAL000/?utm_source=ig_web_copy_link&igshid=MzRIODBiNWFIZA) (accessed Sep. 12, 2023).
- [2] “Baja ETS on Instagram: ‘Rencontre d’information le 21 septembre au local D-2027! il n’est pas trop tard pour joindre l’équipe de baja ets. tous les programmes sont les bienvenues. Au Plaisir de vous rencontrer’” Instagram, [https://www.instagram.com/p/Ciu68AvuA\\_X/?utm\\_source=ig\\_web\\_copy\\_link&igshid=MzRIODBiNWFIZA](https://www.instagram.com/p/Ciu68AvuA_X/?utm_source=ig_web_copy_link&igshid=MzRIODBiNWFIZA) (accessed Sep. 12, 2023).
- [3] J. C. Dixon, “12 - Double Arm Suspensions,” in *Suspension geometry and computation*, Chichester, U.K.: Wiley, 2009
- [4] J. Reimpell, H. Stoll, and J. W. Betzler, “1 - Types of Suspension and Drive,” in *The Automotive Chassis: Engineering principles*, Warrendale: SAE International, 2001
- [5] S. V. Dusane, M. K. Dipke, and M. A. Kumbhalkar, “Analysis of steering knuckle of all terrain vehicles (ATV) using finite element analysis,” IOP Conference Series: Materials Science and Engineering, vol. 149, p. 012133, 2016. doi:10.1088/1757-899x/149/1/012133

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- [6] I. Hiremath, A. Nalawade, and J. Patil, "Design and development of front suspension system for an off-road vehicle," International Journal of Research in Engineering, Science and Management, <https://journal.ijresm.com/index.php/ijresm/article/view/79> (accessed Sep. 14, 2023).
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