Robotic Ankle Exoskeleton

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Project Description

- Partnered with Dr. Lerner and NAU's Biomechatronics Lab to design a lower extremity exoskeleton that aids the motion of walking.
- \cdot Budget \$3800.00, team-fund \$380
- Reduce metabolic power, increase ankle torque
- Designed for people with Cerebral Palsy and similar disabilities
- previous trial "did not observe a significant group level benefit relative to walking without the device. However, [they] did observe a marked benefit for [their] more impaired participants"[1]
- We have reiterated parts of their design (footplate, calf cuff, rod)
- Design A cover for the chain to pulley system
- Some differences (bracket, chain to cable, pulley)
- Focus was on the mechanical structure: how would the components interact (cable-rod, pulley-rod, bracket-rod cut outs)

Figure 1: Exploded View of Design

Project Description

Deliverables

- Create design concepts
- Develop SOLIDWORKS CAD model
- Design mechanical components
- Fabricate/machine parts
- Assemble the ankle exoskeleton prototype
- Reiterate previous bracket design
- Design a chain/pulley cover

Success Metrics

- Budget
- Parts mesh without interference
- Lightweight
- Ease of use
- Complete exoskeleton
- Size is unobtrusive to daily life

Design Requirements

Customer Requirements

- **CR1** Lightweight
- **CR2** Ergonomic- Human Centered **Design**
- **CR3** Durable
- **CR4** Economical or Cost Effective
- **CR5** Low profile- nonobtrusive to daily life
- **CR6** Have a chain to pulley system

Engineering Requirements

- **ER1** \$3,800.00 budget
- **ER2** Range of motion should be 45 degrees in either direction (resting is 90)
- **ER3** Weight < 1 kg per leg
- **ER4** Cannot extrude from the body more than 10 cm
- **ER5** Lifetime of 100,000 steps
- **ER6** Time to take on/off (<60 s)

Figure 2: House of quality for ankle exoskeleton

Benchmarking

Figure 3: Fully Active Chain and Sprocket Ankle Exoskeleton for Rehabilitation Assistance

Figure 4: Robotic ankle exoskeleton by H3

Figure 5: Ultra-lightweight and versatile untethered robotic ankle exoskeleton

Literature Review

- **SHIGLEY'S MECHANICAL** ENGINEERING DESIGN [2]
- Gear Train: Gear Ratios, Torque, and Speed Calculations [3]
- Fundamentals of modern manufacturing: materials, processes, and systems [4]
- Usability and performance validation of an ultra-lightweight and versatile untethered robotic ankle exoskeleton [5]
- Functioning 'mechanical gears' seen in nature for the first time [6]

Literature Review: Avila Literature Review: De Korte Literature Review: Green

- Mechanics of Materials [17]
- Materials Science and Engineering: An Introduction [18]
- Design Optimization [14]
- Design of a Passive Gait-Based Ankle-Foot Exoskeleton [15]
- Untethered Robotic Ankle Exoskeleton [16]
- **Material Selections [19]-[30]**

- Engineering mechanics: Dynamics. [7]
- Biomechanics of Movement: The Science of Sports, Robotics, and Rehabilitation. [8]
- Biomechanics of the ankle [9]
- Foot Biomechanics During Walking and Running[10]
- A REVIEW OF GAIT CYCLE AND ITS PARAMETERS [11]
- "What Is Cerebral Palsy?" [12]
- "Biomechanical and Perceived Differences between Overground and Treadmill Walking in Children with Cerebral Palsy [13]

Mathematical Modeling

Cross Section Choice

 $T = F * r$

Torque is produced from a force acting perpendicular to a body at a specified length

> $I =$ 1 2 $M * r^2$

Moment of inertia for a solid rod

$$
I = \frac{1}{2}M(r_1^2 + r_2^2)
$$

Moment of inertia for a hollow rod

Comparison of inertias revealed that a hollow tube has a higher strength: weight ratio than a solid rod, and would be able to handle torque better

Stress Experienced in the Bracket

- Max Torque Generated during walking test 36 NM
- Diameter of the pulley (.08m) sprocket (.01m)
- Found the Gear Ratio D_{out}/D_{in}
- Found the torque at the Sprocket
- Force acting on the Bracket (880 N)

$$
\frac{\tau_{out}}{\tau_{in}} = Gear \text{ Ratio}
$$

$$
F = \frac{\tau}{R}
$$

Bolt/Rivet or Epoxy

- $F_v = 880 N \sin(45^\circ) = 748.8 N$
- Factory of Safety $(FoS) = 2$
- Using trial and error
- Bolt $d = 9mm$
- Shearing force of bolt = $11.75 \frac{N}{mm^2}$
- Ultimate allowable stress of aluminum is 155 N/mm^2
- Epoxy Shear strength $= 6.89 137$ MPa

Mathematical Modeling – Material Selection

Sprocket

Gear teeth bending stress (Lewis Equation):

 $\sigma_t =$ $W_t P_d$ *FY*

Diametral Pitch:

 $Dp =$ Teeth Pitch

Tangential Load:

$$
W_t = \frac{2T}{Dp}
$$

Variables:

- Pd = Diametral Pitch= 3.75mm
- $T = Torque = 36$ NM
- Teeth $= 30$
- $F =$ Face Width = 11mm from chain width
- $Y =$ Lewis Factor

Gear

- Using the Lewis equation, the tooth bending stress = 828 Mpa
- The allowable stress for various materials
	- Steel=840 MPA
	- \cdot PLA= 37 MPA
	- Aluminum= 290 MPA
- Steel can be used for the sprocket material

Bracket

Aluminum 6061-T6 is the best material to use in this application

Table 1: Table of the various materials

Footplate Thickness

- Dr. Lerner will provide a carbon fiber footplate
- Calculate the needed thickness of the footplate
- $\sigma =$ \overline{F} \overline{A}
	- $F = force$ exerted by user
	- $A =$ to the surface area of the foot
	- \bullet σ = normal stress
- $t = \sigma$ \boldsymbol{L} $\mathcal{S}_{0}^{(n)}$
	- $L = length of footnote$
	- $S =$ allowable stress of material

Assuming the user is an average 14-year-old male Mass: 60 kg Foot length: 24.45 cm Foot width: 9.65 cm

 $S = 3.5$ GPA or $3.5*10^{9}$ Pa

$$
F = mgh \rightarrow F = (60 \text{kg})(0.5) \left(9.81 \frac{m}{s^2}\right) \rightarrow
$$

$$
F = 294 \text{ N}
$$

$$
\mu = 0.5
$$

(*friction coefficient of shoe against ground*)

 $t = 8.7 * 10^{-4}$ mm

Functional Decomposition

Black Box Model

Figure 6: Black Box model

Functional Decomposition

Figure 7: Functional Decomposition

Concept Generation

Figure 8: Morphological design matrix

Concept Selection

Figure 9: Pugh Chart

Table 2: Decision Matrix

• Design 1 was rated the highest

Figure 10: CAD of finalized bracket design

Gantt Chart

ME 476C: Robotic Ankle Exoskeleton

• The team utilized a Gantt chart to stay on track for the entire project

Figure 11: Snapshot of the Gantt chart used for the entire project

Budget

• Our Budget was \$3800, \$3785.53 was spent

Table 3: Budget

Self-Funding Budget

Table 4: Self-funded Budget

- We need to fundraise 10% of our client Budget (380)
- Which was done by selffunding both of our initial prototypes and purchasing some final parts
- We fundraised \$392.29

Manufacturing

• For this device we only needed to Manufacture 4 Parts

Table 5: Bill of Materials (Manufactured)

Purchasing

Table 5: Bill of Materials (Purchased)

1st Initial Prototyping

- How will our bracket Interface with our rod and if there will need to be any adjustments to the current tolerances?
- Which bracket design will be the most stable and rigid?
- To answer these questions, we made two physical prototypes of both bracket designs which our team are considering using.

Figure 12: Bracket design 1 Figure 13: Bracket design 2

Answers and Things we Learned

- •Both designs were stable enough to work as a design
- The tolerance on one of the brackets needs to be adjusted since the piece that goes into the rod is a little difficult to put inside the rod.
- •One valuable thing that we learned was that we need to adjust our design to have the motor placed lower on the rod, so that we can make a smaller hole in the rod for the pulley.
- •To achieve this, we plan on combining out two designs.

2nd Prototype

- How will our complete design mesh together and will there need to be any changes to tolerances?
- Will our chain to pulley system work as designed or will we need to make changes?

Figure 14: 2nd 3D printed Prototype

Main Things Learned

- Difficultly with attaching the motor to the motor bracket cause of the small clearance
- Need to remove links in our chain to help reduce unnecessary weight
- •Tolerance adjustments for bearings holes, because some caused difficulty with the hole being too small or too large

Second Semester Changes

• Bracket design was changed with the guidance of our Client

Figure 15: Updated spark-plug motor bracket

• A cover was designed to reduce possibility of injury due to pinch points

Figure 16: Cover design

Final Design

Figure 17: Final assembled exoskeleton

Video of a user walking in the completed exoskeleton

Ex1 – Weight Test

Purpose: Leg must weigh ≤1000 grams

Anticipated: SOLIDWORKS Analysis = 808.85 grams

Results: 793 grams

Ex2 – Range of Motion Test

Purpose: Leg be able to move 45º in either direction

Anticipated: SOLIDWORKS Analysis $= 128.54$ ^o in either direction

Results: 47º Forward, 49º Backward

Figure 18: Image of weight test **Figures 19-20:** Images of range of motion test Figure 21: Image of measurement test

Ex3 – Measurement Test

Purpose: Leg should not extend more than 10 centimeters from the user

Anticipated: SOLIDWORKS Analysis $= 4.95$ centimeters

Results: 5 centimeters

Testing

Ex4 – Cost Analysis

Purpose: One leg must cost ≤ \$1,900

Results: \$1786.57

Ex5 – Durability Test

Purpose: Leg needs to withstand $\geq 100,000$ steps

Results: 375,200 steps

Figure 22: Fatigue Analysis

Ex6 – Time Test

Purpose: User should be able to put on leg in ≤ 60 seconds

Results: 3 Tests (22s, 18s, 20s); 20 second average

Testing Results

Future Work

- Redesign Cover to remove excess material and help with weight reduction.
- Add Cable caps to help reduce fraying of the steel wire rope
- Do additional testing for material selection of our bracket
- Outdoor Walking test to see if dust and debris effect devise
- Long term test +30 min

Any Questions?

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