

# SAE Baja: Engineering Analysis Suspension and Steering

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# Overview

- Introduction
- Analysis of Suspension Components
  - Geometry
  - Body Forces
  - Suspension Materials
- Analysis of Steering Components
  - Rack and Pinion
  - Tie Rods
- Gantt Chart
- Conclusion

# Introduction

Client: Professor John Tester

- Explain Suspension/Steering Analysis
  - Dimensions
  - Geometry
  - Body Forces acting on System

# Analysis of Suspension Components

## Front Suspension

- We started by assuming the following specs
- 20" front frame width
- About 12" A-arms
- 205-80-12 tire size

# Analysis of Suspension Components

## Kingpin Angle and Scrub Radius

### ESTABLISHING KNUCKLE AND BALL JOINT LOCATION

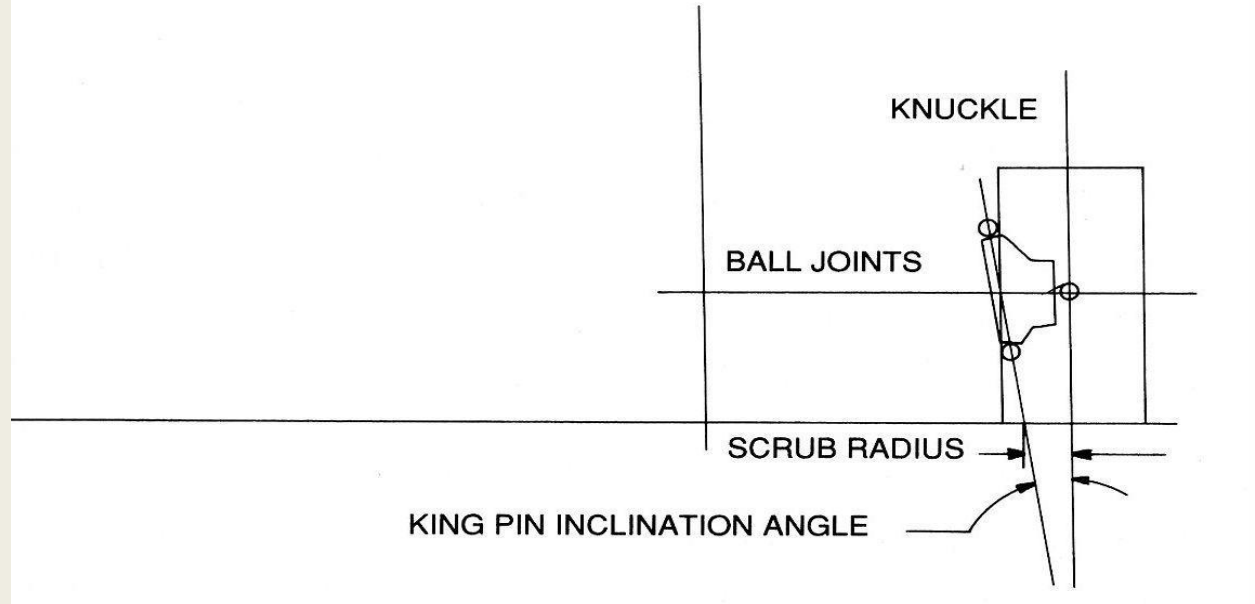


Figure 1: Knuckle & Ball Joint

Source: Chassis Engineering

# Analysis of Suspension Components

## **Ideal Scrub Radius**

- The ideal scrub radius is zero
- This minimizes steering effort.
- This minimizes twisting forces from bumps and cornering forces.

# Analysis of Suspension Components

## King Pin Angle

- Typical Kingpin Angles are between 5-10 degrees.
- Excessive king pin angle causes the tire to flop from side to side as it is steered.

# Analysis of Suspension Components

## **Other Considerations**

- The general Compression to Droop ratio is about 60% to 40%
- We want moderate camber loss on compression.



# Analysis of Suspension Components

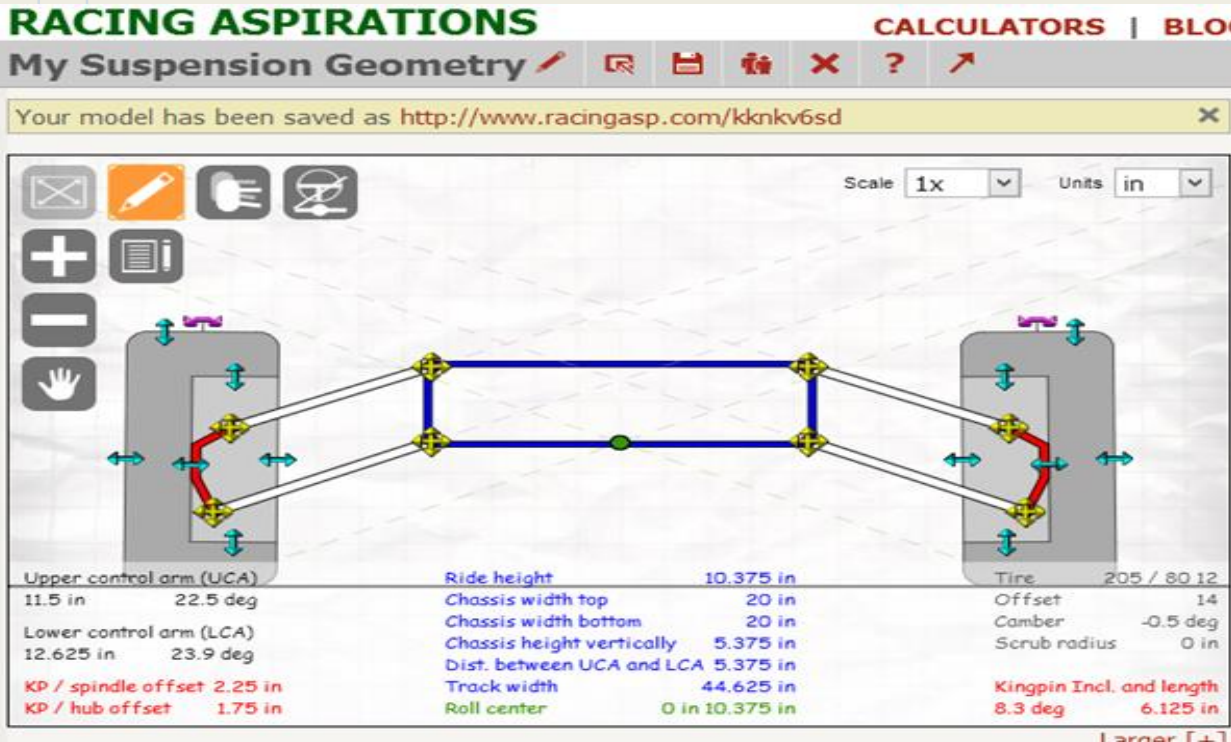


Figure 2: Suspension Analysis

# Analysis of Suspension Components

RACING ASPIRATIONS

CALCULATORS | BLOG

My Suspension Geometry

Your model has been saved as <http://www.racingasp.com/kknkv6sd>

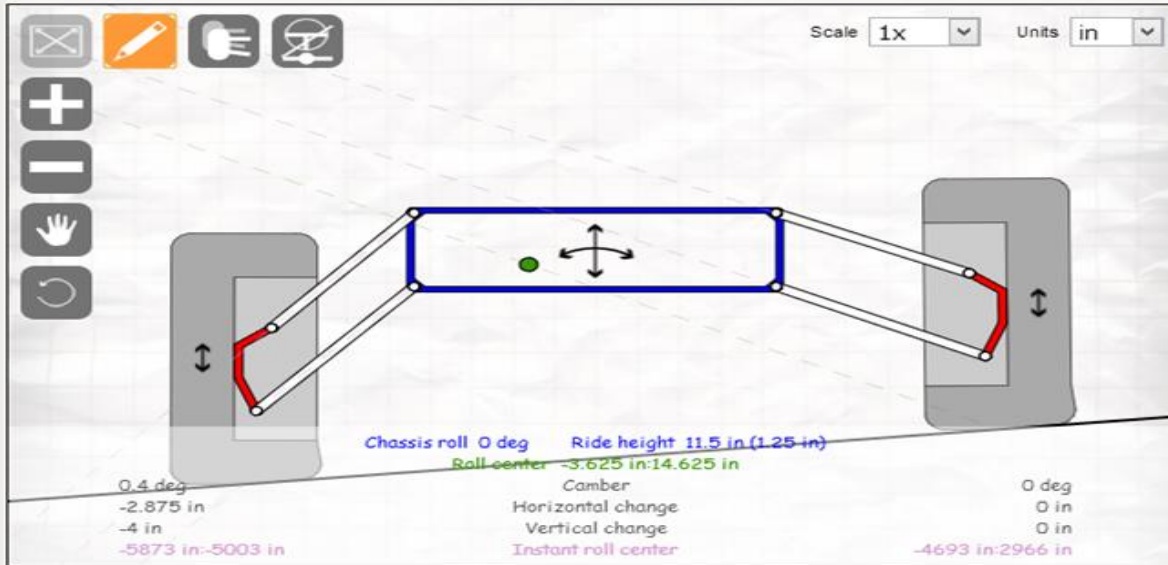


Figure 3: Suspension Analysis

# Analysis of Suspension Components

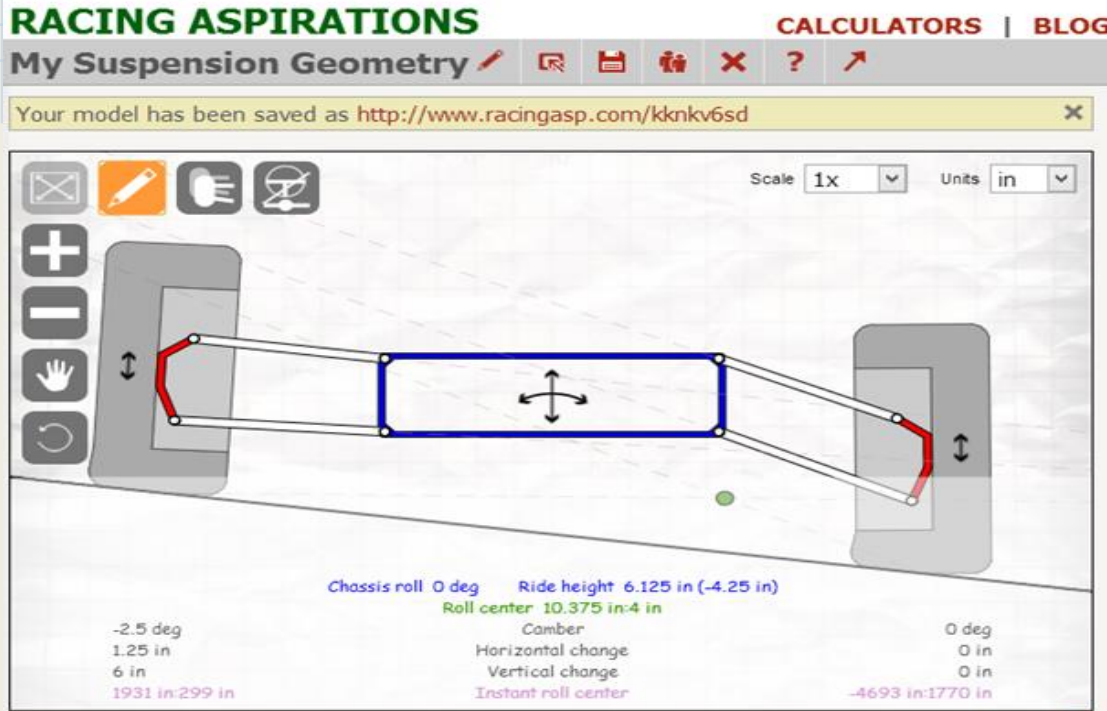


Figure 4: Suspension Analysis

# Analysis of Suspension Components

## Drop Test Assumptions:

- $F_i$  = Force of impact
- $F_s$  = 500 lb Weight
- $h$  = 6 ft Drop Height
- $K$  = 160 lb/in Spring rate constant (using shocks from Polaris RZR 570)
- Force assuming worst case landing on one wheel
- $F_i = F_s + \left( (F_s)^2 + 2 * K * 12 * F_s * h \right)^{\frac{1}{2}}$  (Source SAE Brasil)
- $F_i$  = 1022.53 lb

# Analysis of Suspension Components

## Shock Mounting

### Shock Mounting

- In order to keep from having an over engineered shock, the length away from the pivot and the angle away from vertical
- The longer lever arm on the shock, the stronger it has to be
- The larger the angle away from vertical the shock is mounted, the less effective it is

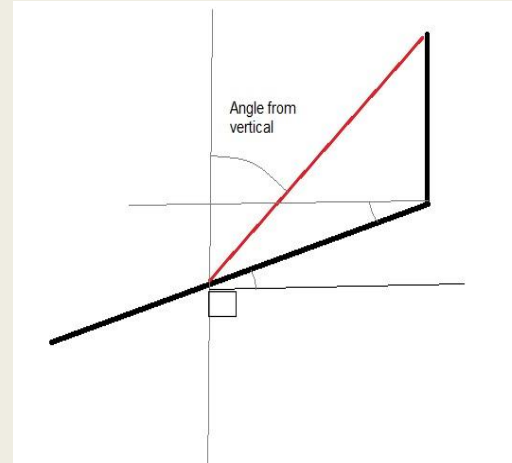


Figure 5: Shock Mounting

# Analysis of Suspension Components

## Shock Mounting (cont.)

- From Matlab code, from a given mounting distance, the angle away from vertical and the shock length are found
- Compromising between the mounting position and the angle away from vertical, the team decided on mounting 9 inches away from the frame
  - Shock length is 11.5 inches
  - The angle away from vertical will be 39 degrees
- Shock will have to be ~1.5 times stronger compared to a vertical shock

# Analysis of Suspension Components

## Suspension Materials:

Table 1: Suspension Material

Material	Properties					
	Sy (ksi)	Ts (ksi)	E (ksi)	$\rho$ (lb/in <sup>3</sup> )	G (ksi)	$\nu$
AISI 1018 (CD)	54	64	29000	0.284	11600	0.292
AISI 4130 (normalized)	63	97	29700	0.284	11600	0.292

# Analysis of Suspension Components

## Why AISI 4130?

- High tensile strength
- High yield strength
- Meets SAE rules for approved materials
- Maintains physical properties after welding
- Widely used in off-road designs



# Analysis of Suspension Components

## 3-link Trailing Arm

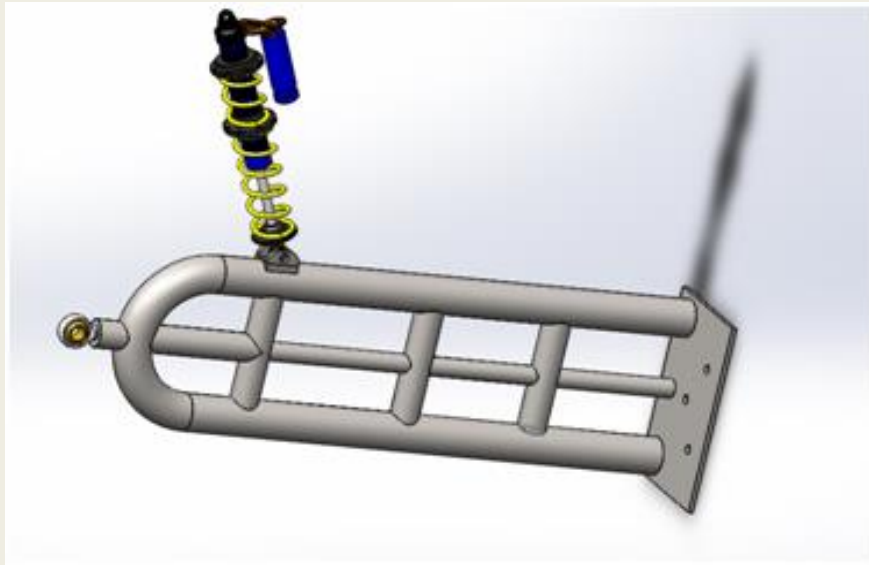


Figure 6: Trailing Arm

- Shown without additional links and knuckle
- Material: AISI 4130 normalized
- $L = 28$  in
- $OD = 1.5 - 2.0$
- $t = 0.083$
- End plate attaches to knuckle
- Assuming square shafts, attached to wall

# Analysis of Suspension Components

## Drop Test Results:

**Table 2: Drop Test results**

Force Location (in)	Force (lbf)	Yield Strength (ksi)	Deflection (in)
25	6600	97	0.000069
26.5	5543.34	97	0.125

# Analysis of Steering Components

## Steering System

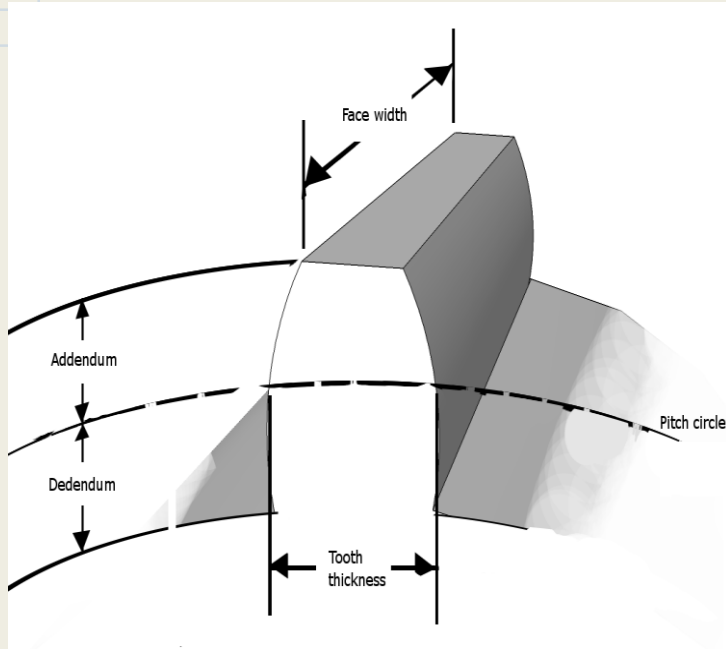
- Rack and Pinion
  - Calc. for designing
    - Most likely buy
  - Gives ideas of material selection
- Dimensions for Pinion
  - Several Assumptions (Driver 0.1-10 lbf)
  - Rack teeth => pinion turns 180 deg max

# Analysis of Steering Components

**Table 3: Dimensions of Pinion and Rack**

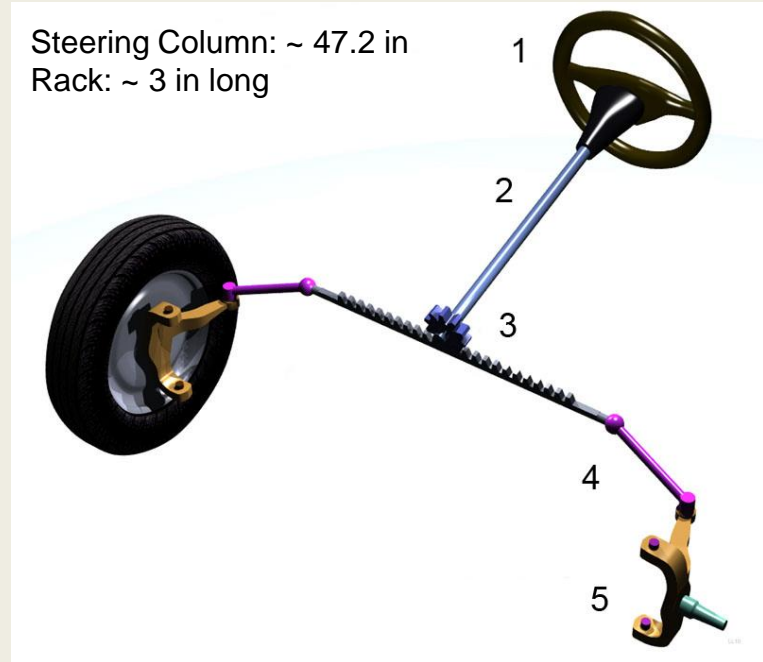
	teeth number	face width (in)	bending stress (kpsi)	radii for pitch circle (in)	radii for base circle (in)	Adden (in)	Dedden (in)
pinion	20	0.74	0.04 - 3.9	0.787	.739	0.078	0.098
rack	20	0.74	-	inf	inf	0.078	0.098

# Analysis of Steering Components



**Figure 7: Pinion Geometry**

Source: <http://psas.pdx.edu>



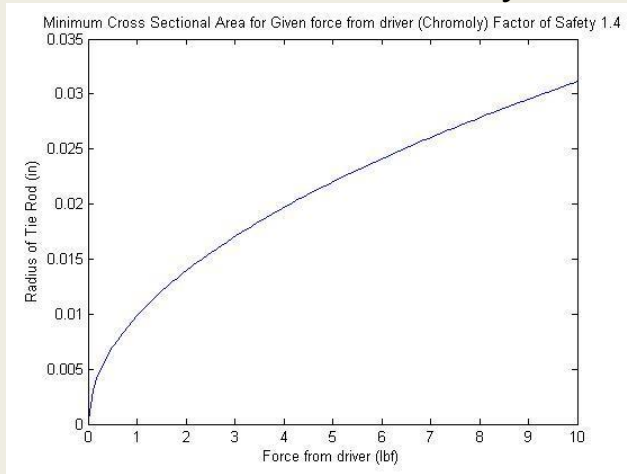
**Figure 8: Rack & Pinion**

Source: wikipedia

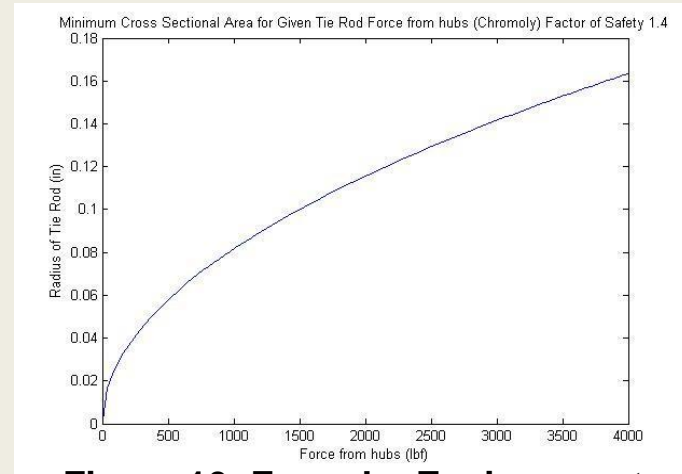
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# Analysis of Steering Components

- Analysis of Tie Rod (Chromoly AISI 4130)
  - Force exerted by environment
  - Force exerted by driver



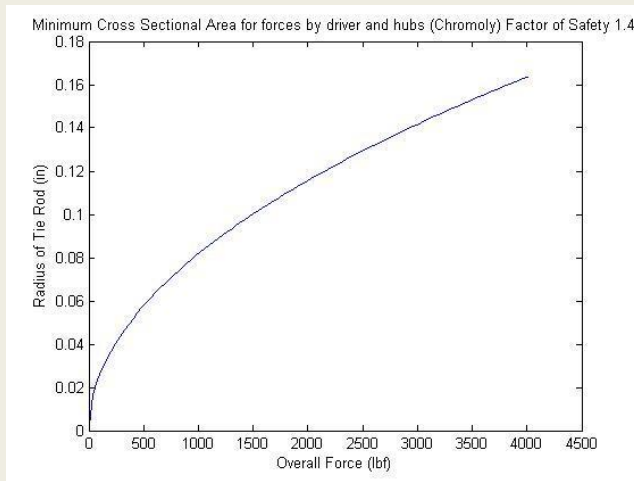
**Figure 9: Force by Driver**



**Figure 10: Force by Environment**

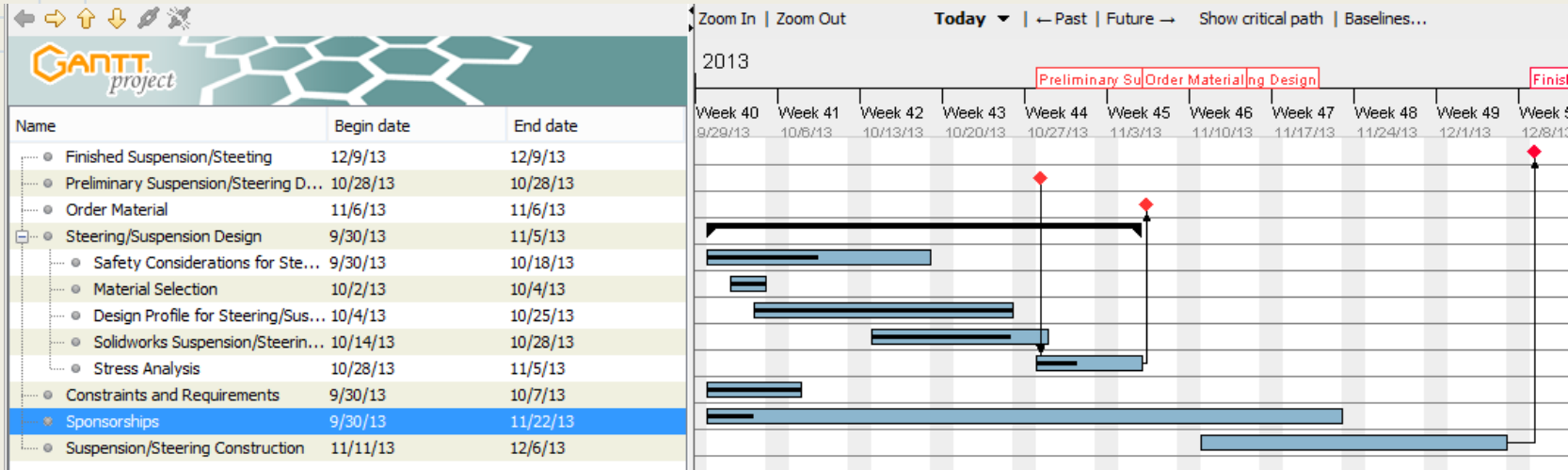
# Analysis of Steering Components

- Force exertion by hubs and driver (Chromoly AISI 4130)
  - Tie-Rod radius increases as Driver and Hub Force increase together



**Figure 11: Overall Force**

# Gantt Chart





# Conclusion

- Suspension Component Analysis
  - Spring & Joint Mounting Locations
  - Calculated Deflections at Maximum Drop Force
- Steering Component Analysis
  - Gearing of Rack and Pinion at given situations
  - Tie rod thickness at given situations

# References

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- Adams, Herb. *Chassis Engineering*. Los Angeles: HP, 1993. Print.
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