

Human Powered Vehicle Engineering Analysis

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Overview

- Project Description
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 - Frame
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Project Description

- ASME Human Powered Vehicle Challenge
- Clients – Perry Wood & ASME
- “Design a human powered vehicle that can function as an alternative form of transportation.”
- Objectives
 - High Speed
 - Aerodynamic Drag
 - Maneuverable

Frame

- 3 Sections: Center Tube, Outriggers, Roll Bar
- 5 Configurations
- Analyzed stresses, deflection, and weight
 - Hand calculations
 - Finite Element Analysis (FEA)

Table 1- Hand Calculation Results






Configuration	 1.5"ODx.058"ST	 2"ODx0.125"AL	 1.5"x1.5"X0.125"AL	 1.75"ODx0.125"AL	 2"x1"x0.125"AL
Main Tube Deflection [in]	0.392	0.230	0.342	0.353	0.225
Outrigger Deflection [in]	0.183	0.107	0.159	0.165	0.105
Main Tube Lateral Deflection [in]	0.196	0.115	0.171	0.176	0.356
Outrigger Lateral Deflection [in]	0.069	0.040	0.060	0.062	0.125
Weight [lb/in]	0.075	0.071	0.066	0.061	0.066
Outrigger Stress [psi]	46598	13077	14593	17551	12813
Outrigger Stress Max [psi]	55917	18961	22473	25448	19732

Figure 1- Square Outrigger Stress

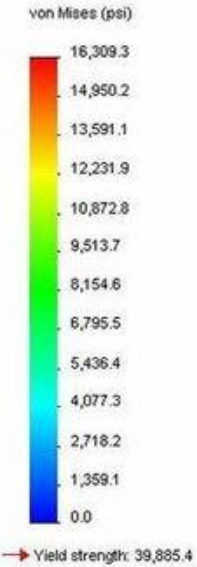
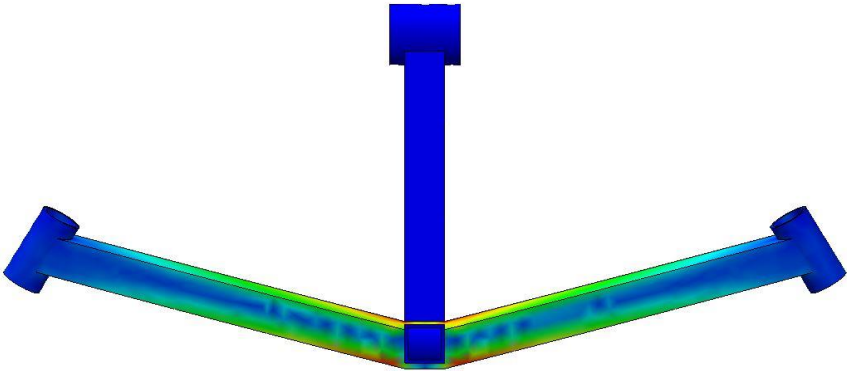


Figure 2- Square Outrigger Deflection

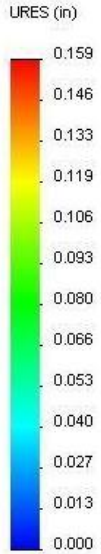
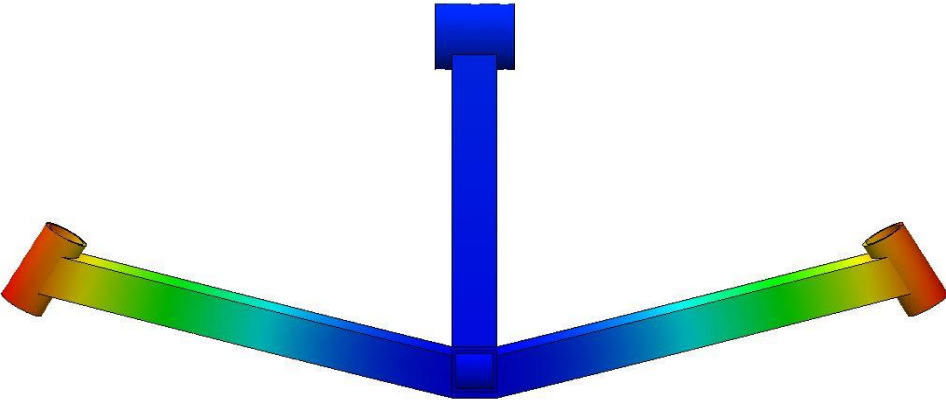


Table 2- Outriggers FEA vs. Calculated Results



Configuration	 1.5x1.5x0.125AL	 1.75ODx0.125AL
Calculated Deflection [in]	0.159	0.165
FEA Deflection [in]	0.159	0.139
Calculated Nominal Stress [psi]	14593	17551
Calculated Max Stress [psi]	22473	25448
FEA Stress [psi]	16309	21897

Figure 3- Top Load Roll Bar Deflection

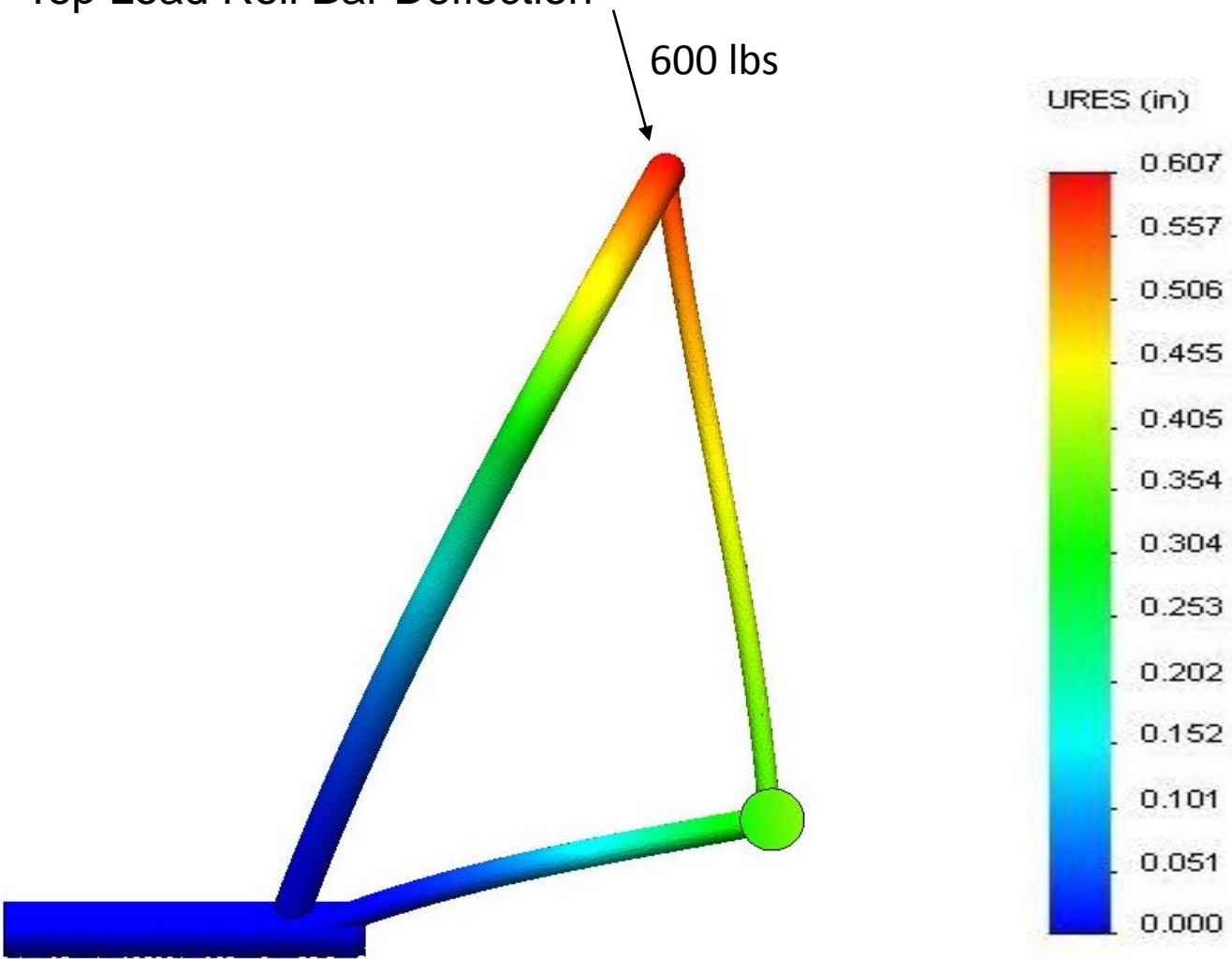
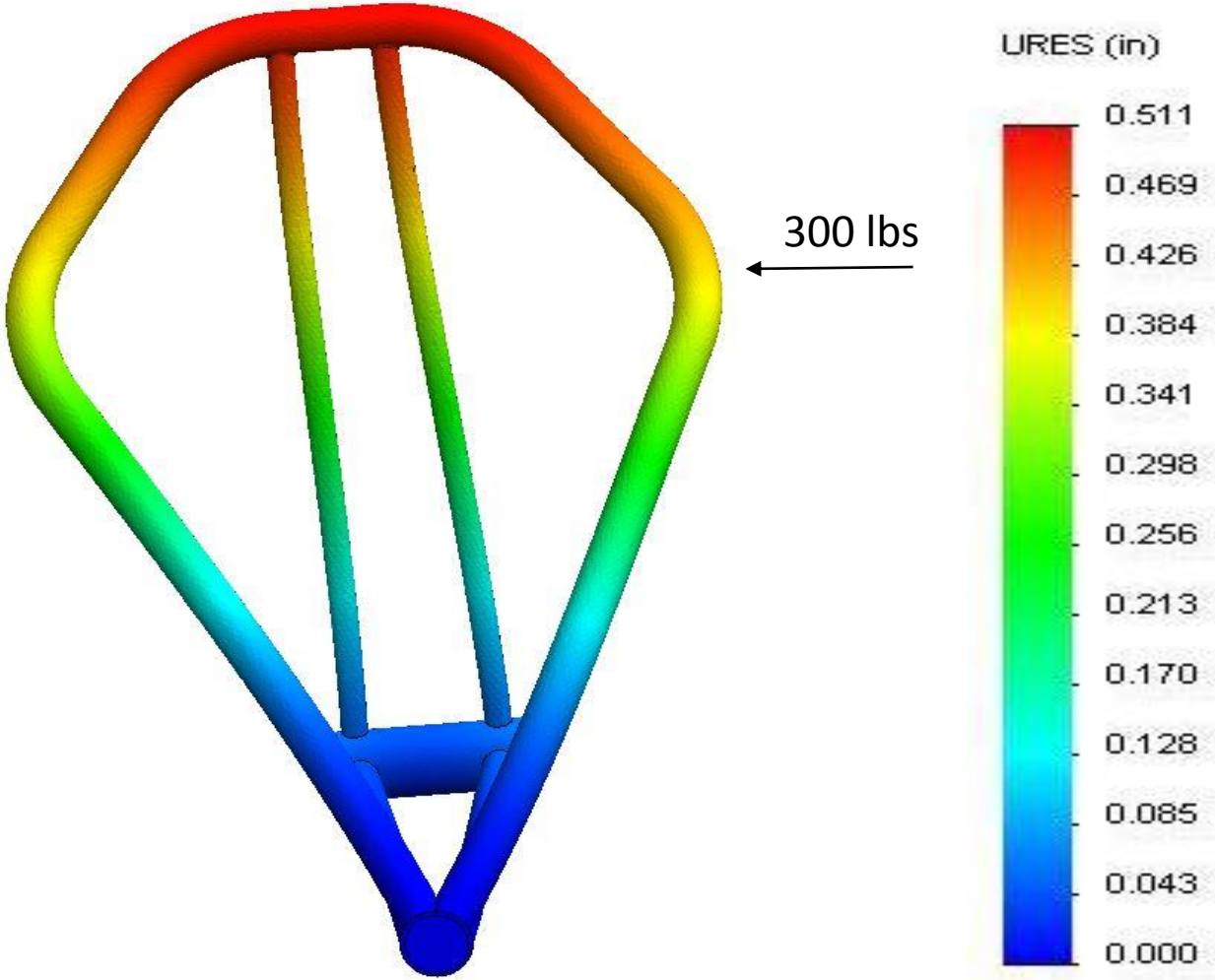


Figure 4- Side Load Roll Bar Deflection



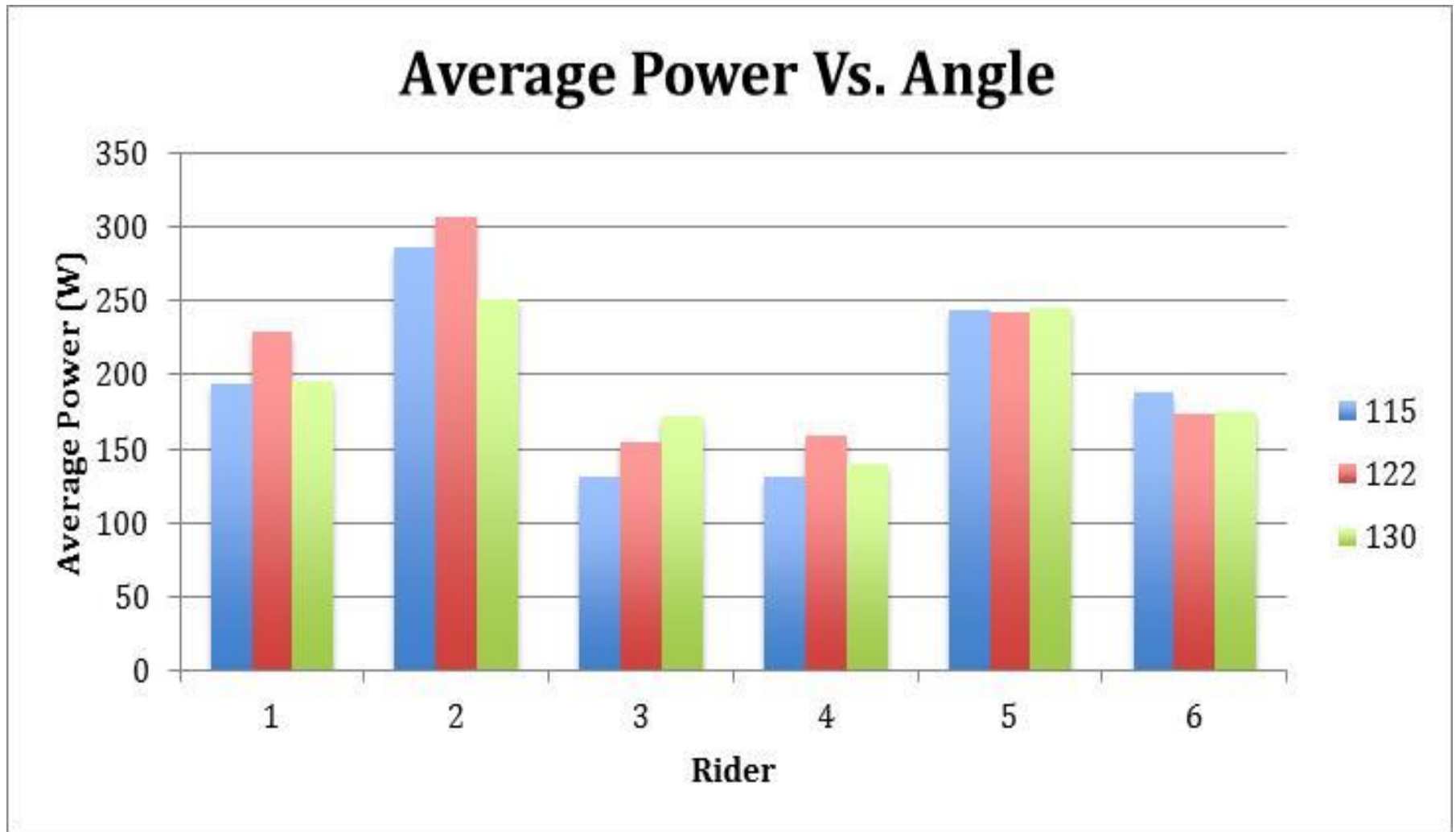
Ergonomics

- Rider Position Study
 - Three test angles
 - 115°, 122°, 130°
 - 1 min. sprint
 - 3 min. endurance
 - Power and cadence (average and max)

Figure 5- Rider Position Angle



Figure 6- Average Power at Various Angles



Fairing

- SolidWorks Flow Simulation
- General dimension changes
 - Length: 96-108 in
 - Width: 18-24 in
 - Height: 33-39 in
- Assumptions
 - Air
 - Temperature
 - Velocity
 - Roughness

Table 3- Coefficient of Drag Comparison

Length (in)	Width (in)	Height (in)	Speed (in/s)	Force (lbf)	Area (in ²)	Cd
96	18	37	704	0.59	681.5	0.038
96	20	37	704	0.51	716.5	0.031
96	22	37	704	0.54	760.0	0.031
96	24	37	704	0.61	803.7	0.033
102	18	37	704	0.41	670.3	0.026
102	20	37	704	0.49	702.1	0.030
102	22	37	704	0.56	753.5	0.032
102	24	37	704	0.51	790.6	0.028
108	18	37	704	0.54	670.5	0.035
108	20	37	704	0.48	701.4	0.030
108	22	37	704	0.43	740.0	0.025
108	24	37	704	0.57	788.4	0.032

Steering

- Steering Geometry
 - Caster Angle
 - Camber Angle
 - Kingpin Angle
- Steering Knuckle FEA
 - Aluminum
 - Chromoly

Figure 7- Caster Angle

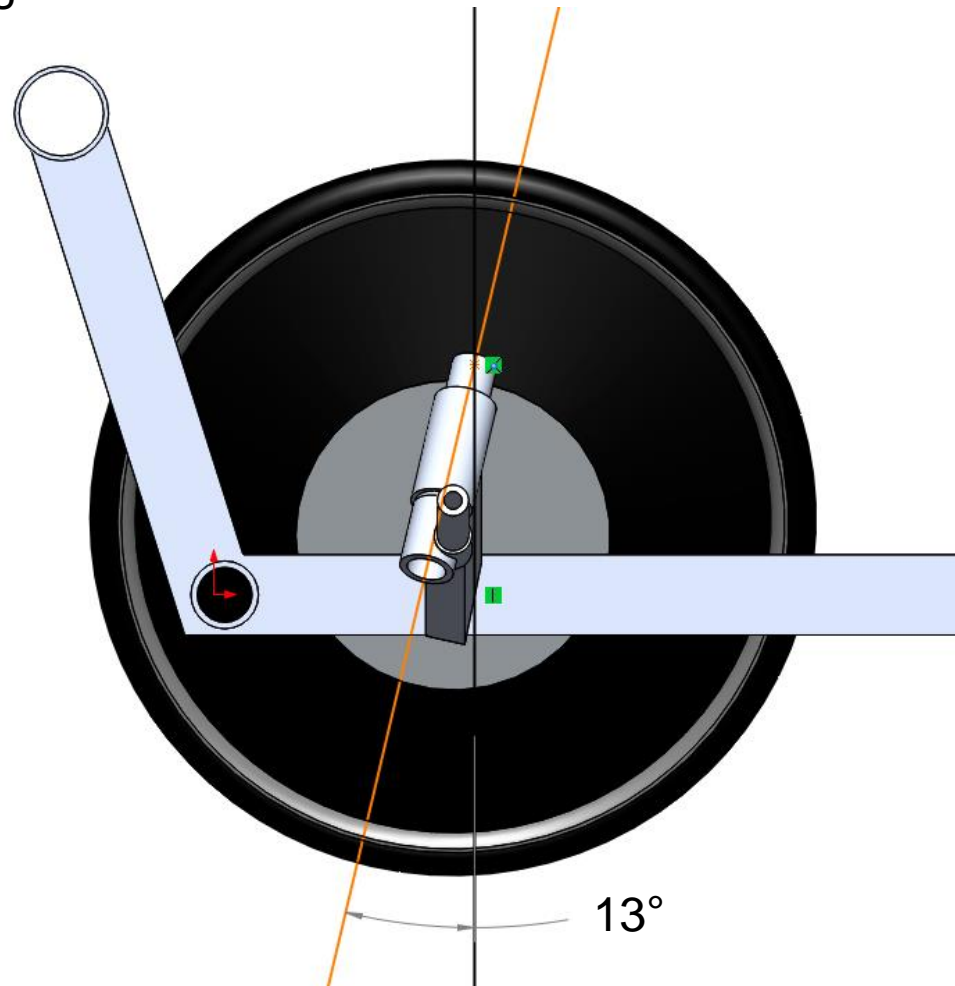


Figure 8- Camber Angle

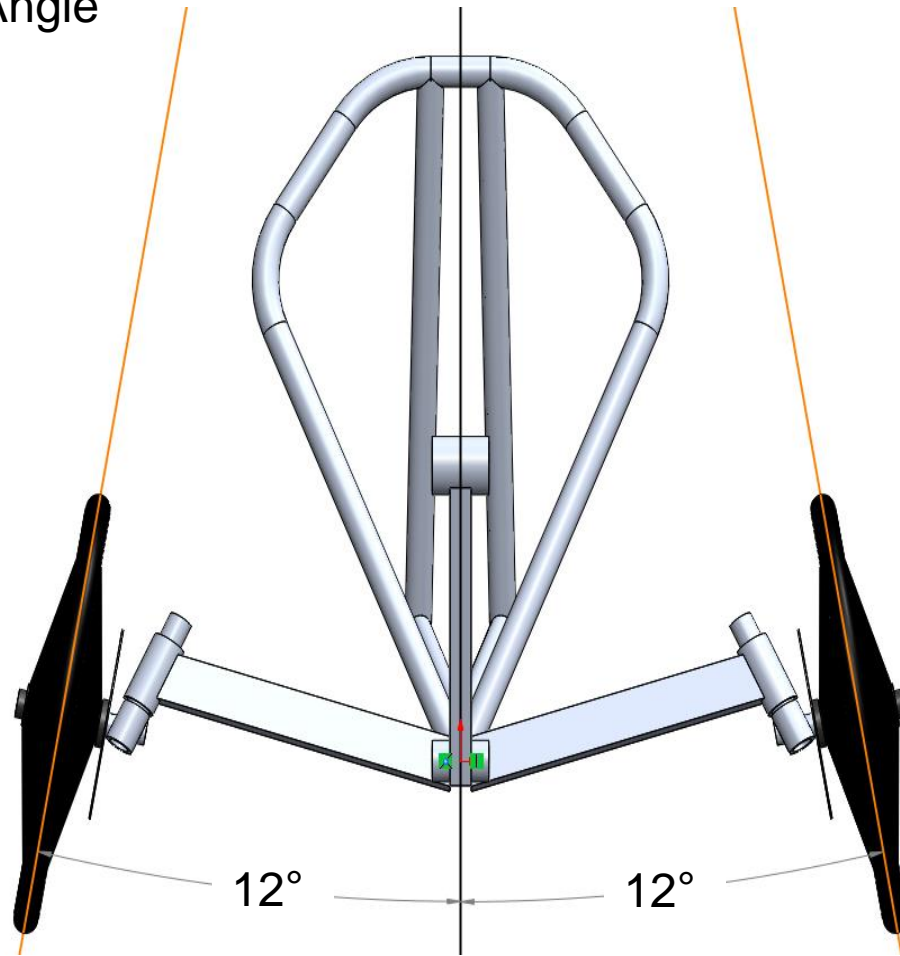


Figure 9- Kingpin Angle

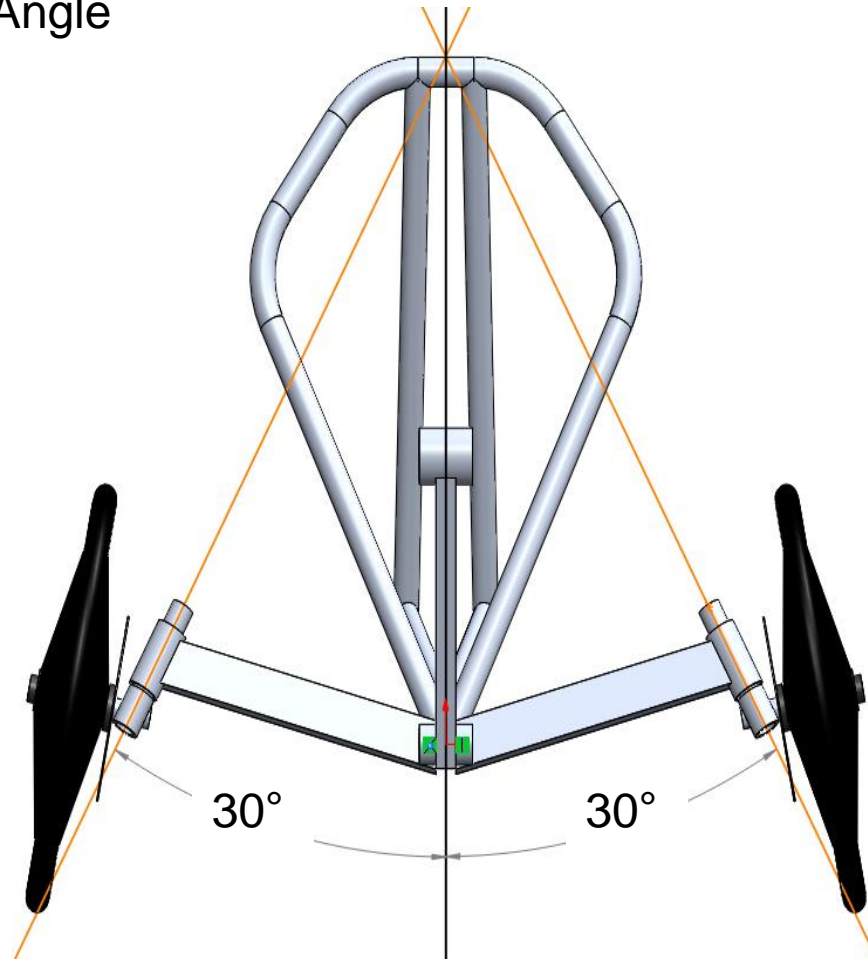


Figure 10- Aluminum FEA

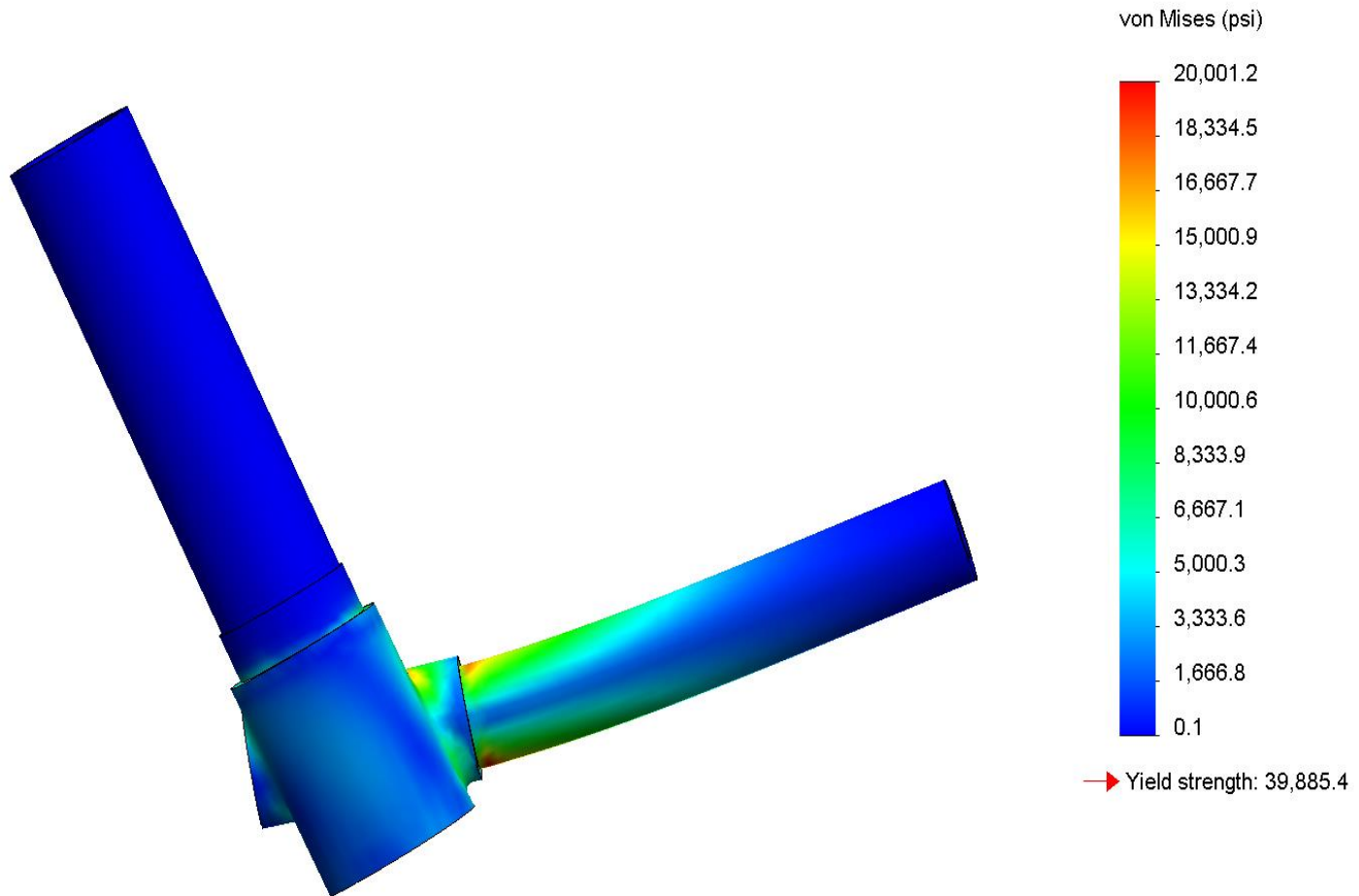
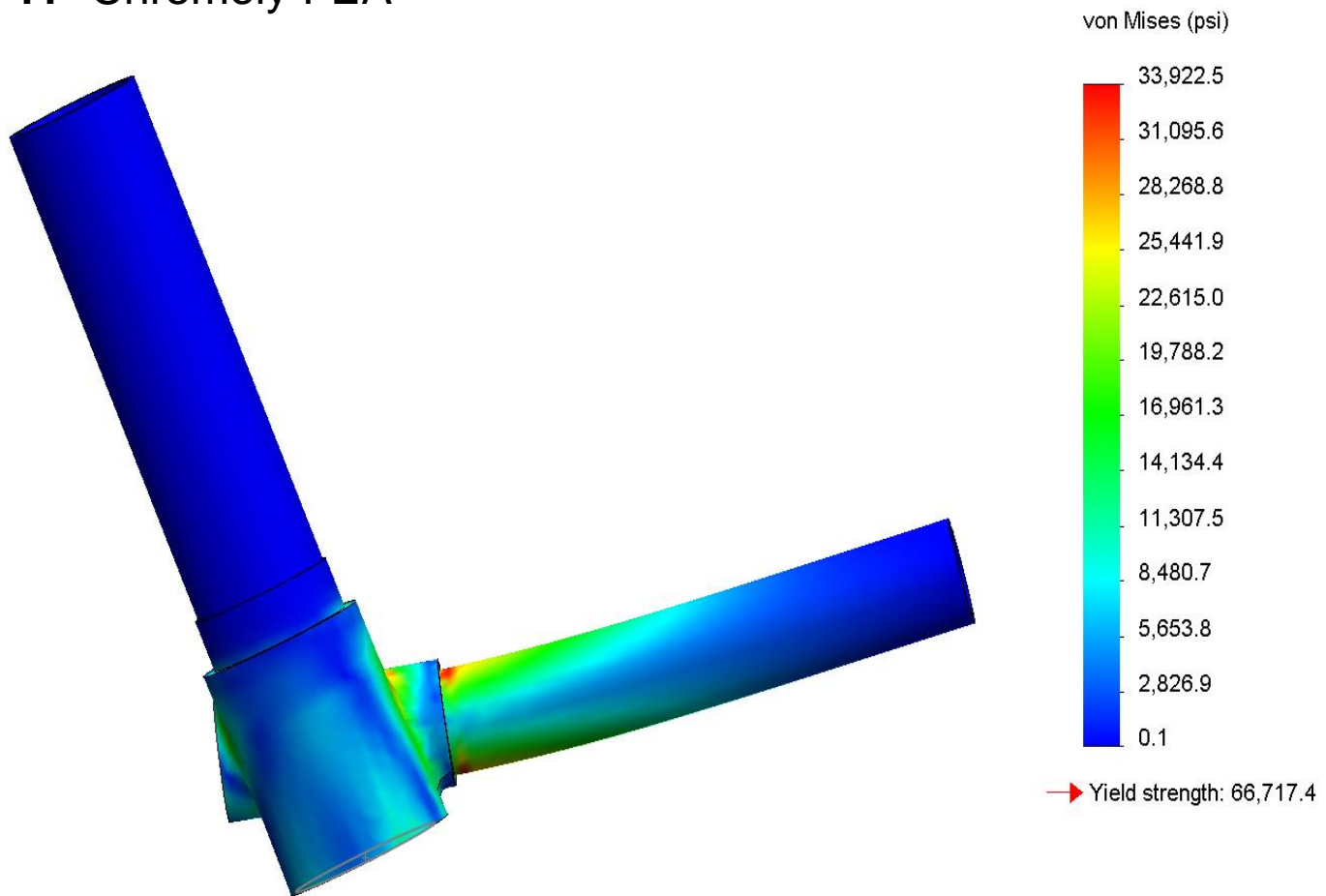


Figure 11- Chromoly FEA



Drivetrain

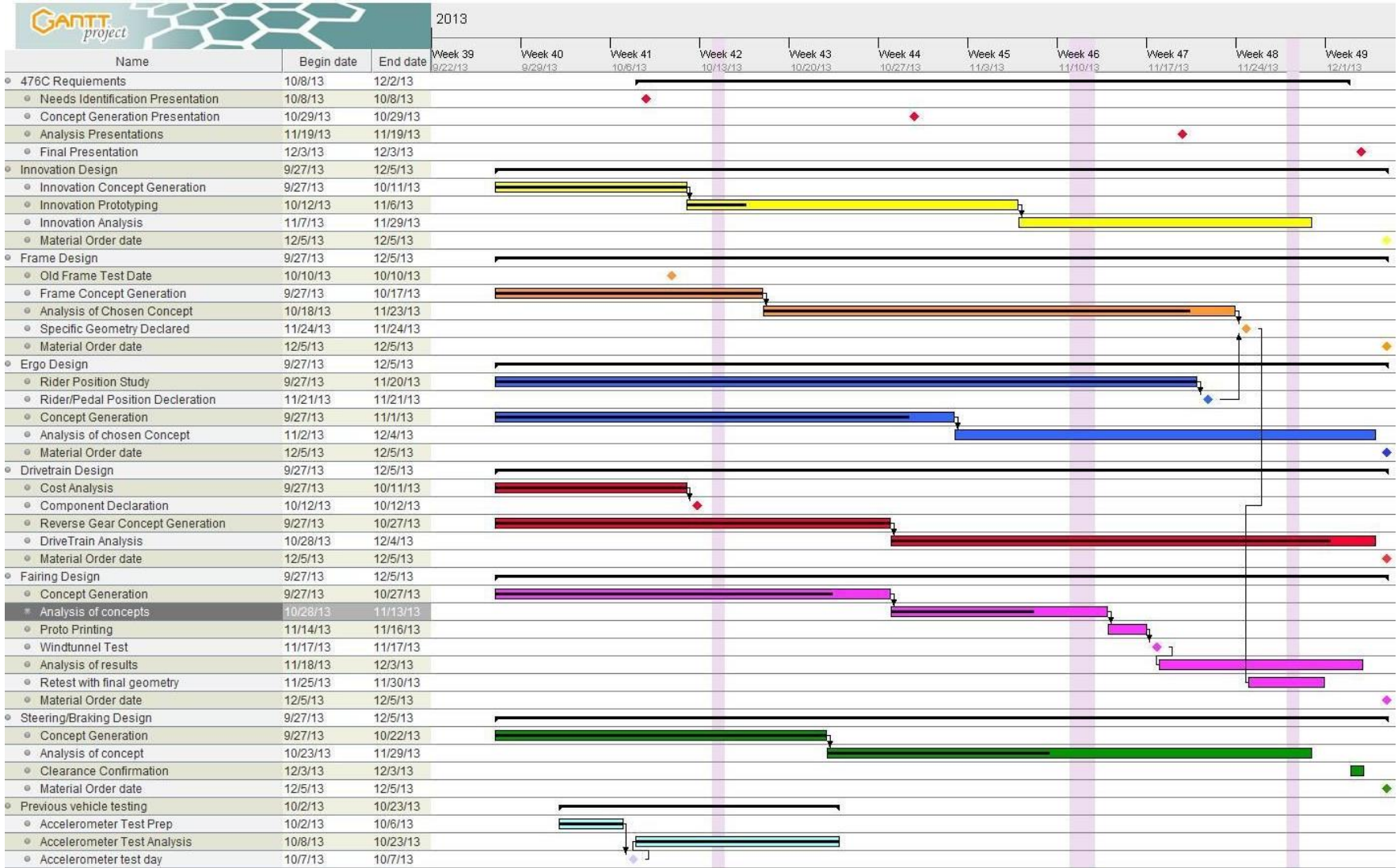
- Reach 40 MPH
- Lowest possible gear ratio
- Cadence values from rider position study

Table 4- Gear Ratios and Speeds

Gear Ratio	Speed at 90 RPM (MPH)	Speed at 110 RPM (MPH)
1.50	10.56	12.91
1.69	11.88	14.52
1.93	13.58	16.60
2.25	15.84	19.36
2.57	18.11	22.13
3.00	21.13	25.82
3.38	23.77	29.05
3.86	27.16	33.20
4.50	31.69	38.73
4.91	34.57	42.25

Project Plan Gantt Chart

Figure 12- Project Plan



Conclusion

- The frame will use 1.5 in x 1.5 in aluminum square center tubing and outriggers to minimize weight and deflections.
- The rider position will be at an angle of 122 degrees for visibility and efficiency.
- The fairing will have an approximate length of 108 in, width of 22 in and height of 37 in to give a minimal coefficient of drag of 0.025.
- The steering knuckle will be made out of aluminum to reduce weight while maintaining a factor of safety of 2.
- The drivetrain will minimize the gear ratio while achieving a max speed of over 40 mph.

References

- [1] R.C. Hibbeler, *Structural Analysis*, New Jersey, Pearson Prentice Hall, 2012
- [2] R. G. Budynas and J. K. Nisbett, *Shigley's Mechanical Engineering Design*, New York, McGraw-Hill, 2011
- [3] Philip J. Pritchard and John C. Leylegian, *Introduction to Fluid Mechanics*, Manhattan College: John Wiley & Sons, Inc., 2011.
- [4] R. Horwitz Author. (2010). The Recumbent Trike Design Primer (8.0) [Online]. Available: http://hellbentcycles.com/trike_projects/Recumbent%20Trike%20Design%20Primer.pdf
- [5] R.C. Hibbeler, *Engineering Mechanics – Statics*, Pearson Prentice Hall, 2010

Questions?